

# Torsion of Rolled Steel Sections in Building Structures

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TORSIONAL STRESSES in a steel framed building are rarely serious enough to require design analysis. Many torsional situations can be disregarded completely. There are conceivable conditions, however, in which torsional loads can produce stresses of sufficient magnitude to require torsional analysis of a framing member.

It is important, therefore, that the structural engineer understand the principles of torsional behavior in rolled steel sections, and be able to recognize those special situations in which torsional loading may be significant to the design.

The purpose of this paper is to provide practical guidance to the designer in the evaluation and analysis of the effects of torsional loading on steel framing members. Design examples illustrate both a "short" approximate method of torsional analysis and a more exact method. A brief review of basic torsional theory and the torsional properties of rolled steel sections are included.

forces, equal in magnitude and opposite in direction. These forces are acting normal to the axis of the member. The usual torsional loading is that of a vertical load  $P$ , which does not pass through the shear center  $S$ , of the cross-section. The distance from load  $P$  to the shear center  $S$  is called the eccentricity,  $e$ . The eccentric moment that induces torsion on the cross-section is  $Pe$ . This moment is called  $M$ . (In this paper  $M$  represents a concentrated torsional moment, and  $m$  represents a uniformly distributed torsional moment induced in a member.)

**Effects of Induced Torsion**—The aforementioned types of torsional loading are the cause of twisting in a member. The effect of this twisting can be threefold in a  $W$ ,  $I$  and  $L$ -section: torsional shear stresses, torsional warping shear stresses and torsional normal (longitudinal bending) stresses. These additive stresses often occur together with shear and normal stresses due to plane bending. Of these torsional effects, the magnitude of the torsional normal stresses is much greater than any of the other torsional stresses. The emphasis on normal stresses resulting from torsion will be discussed later in this paper.

**Internal Equilibrium**—The development of torsional stresses in a cross-section of a member is the result of internal resisting moments, which balance the applied moment  $M$  or  $m$ . For torsional equilibrium, then, the fundamental equation is

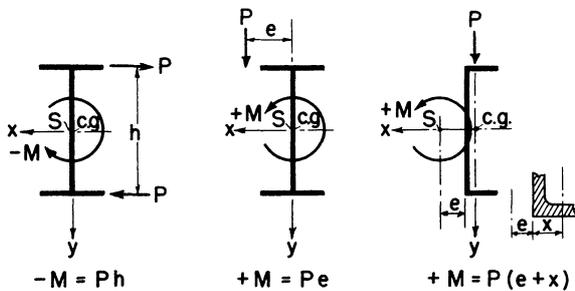


Figure 1

## TORSIONAL BEHAVIOR

**Torsional Loads**—When a structural member is twisted about its longitudinal axis, it is said to be in torsion. Twisting is caused by external forces, moments or a couple acting on the member. Figure 1 shows these types of loads. The couple is composed of two parallel

$$M_t = M_p + M_s \quad (1)$$

where  $M_t$  is the total of the internal resisting moments producing torsional shear,  $M_p$  is a torsional resisting moment called *primary torsion* ( $M_p$  is also known as "St. Venant's" torsion, "pure" torsion and "unrestrained" torsion),  $M_s$  is a *secondary* torsional moment which expresses warping resistance of a cross-section ( $M_s$  is often called "warping" torsion). Actually, Equation (1) is a statement of shear equilibrium, as both  $M_p$  and  $M_s$  produce shearing stresses. It is generally assumed that normal stresses  $\sigma$ , resulting from primary torsion  $M_p$  are negligible, unless the angle of twist is very large.

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### Analogy between Torsion and Plane Bending—

Figure 2 shows a wide flange beam loaded at midspan with a concentrated torsional load  $M$  equal to unity. As shown, the ends of the beam are considered simply framed, AISC Type 2 construction, where it is assumed that the ends of the beam are free to rotate under loading which produces *plane bending*. A web connection consisting of two clip angles may be considered as Type 2 construction (See Fig. 8b). Under *torsional loading*, however, this connection prevents twisting of the beam about its longitudinal axis at the connection, since the web is restrained.

At any cross-section of the beam from **A** to **C** the internal moment  $M_t$  equals  $M_1$ , and from **C** to **B**,  $M_t$  equals  $-M_2$ . It can be seen also that  $M_1a - M_2b = 0$  and  $M_1 + M_2 = M$ , and finally the following expressions

$$M_1 = \frac{b}{l} M \text{ and } M_2 = \frac{a}{l} M \quad (2)$$

An interesting analogous relationship is suggested by Equation (2). The analogy exists between the shear diagram due to plane bending and the moment diagram due to torsional twisting (Fig. 2). If a concentrated load  $P$  is applied at midspan, the shear at each end of the beam will be  $V = P/2$ . In the case of a torsional concentrated load  $M$  at midspan, the end moments  $M_1$  and  $M_2$  are equal to  $M_b$ , and  $M_t = M/2$ .

The moments  $M_1$  and  $M_2$  are determined in the same manner as the reactions at **A** and **B** due to a single load at **C**. It should be emphasized explicitly that this simple, apparently obvious analogy is by no means valid in general. For beams framed with Type 2 construction, the assumption that  $M_t = M_1$  and  $M_2$  at ends **A** and **B** is valid and Equation (2) is directly analogous. However, when beams are framed with Type 1 connections (welded, fully rigid) this assumption is not always strictly valid because of unsymmetrical loading conditions and the redundancy of the supports. Table 1 makes this distinction, and exact values are given when this analogy is not strictly valid. For beams subjected to a uniform torsional loading  $m$ , the same comments and restrictions for concentrated torsional loads apply.

**Influence of  $M_p$** —The torsional resisting moment  $M_p$  is well named “primary”, because it always appears to some extent in a beam under torsion. It is zero in a cross-section only when that cross-section is completely restrained against warping, such as at a fully welded connection. (See Case 5, Table 1.) In Fig. 3b, the cross-section at midspan must remain planar due to symmetry of load  $M$  (note that the angle of twist  $\phi = 0$  at each end, viz., no twisting). Thus, since this cross-section at midspan is completely restrained against warping,  $M_p = 0$  also. Warping is defined as a plane which when acted

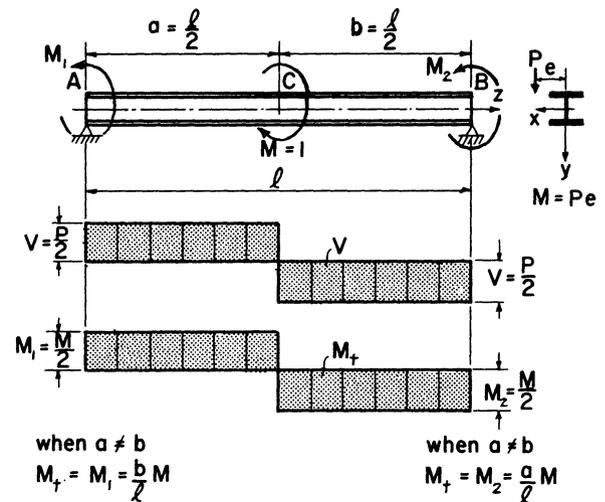


Figure 2

upon by forces, no longer remains planar; it warps out of plane.

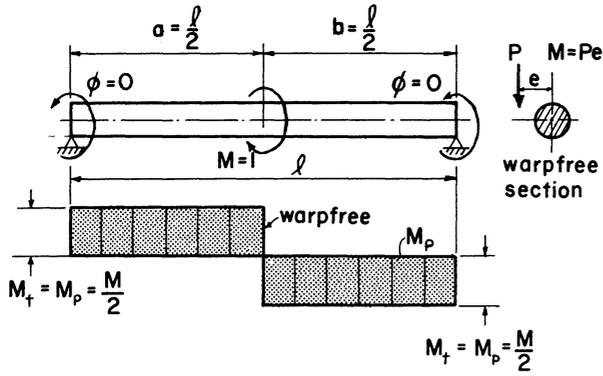
The fundamental equation for primary torsion, for *non-circular* sections, is

$$M_p = GK\phi' \quad (3)$$

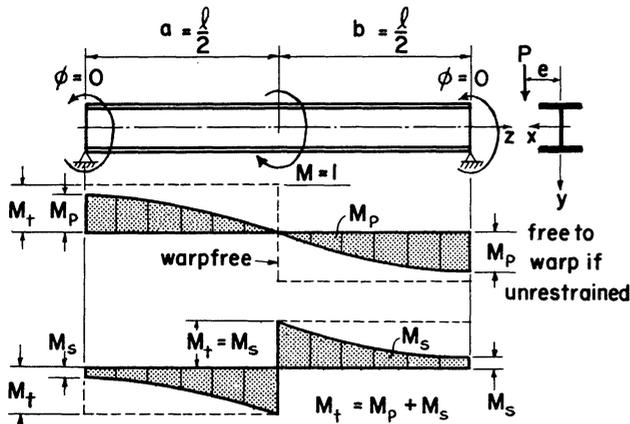
where  $G$  is the shear modulus,  $\phi'$  is the first derivative of  $\phi$  (the angle of twist) with respect to  $z$ , and  $K$  is a torsional constant which describes the *torsional resistance* of a cross-section, and is based upon the geometry of the profile. Values for the torsional constant  $K$  for rolled steel sections have been published by Bethlehem Steel Corp.<sup>1</sup> Reference should also be directed to a recent paper by El Darwish and Johnston<sup>6</sup> covering an accurate calculation of this constant. For cross-sections made-up of *welded plates*, values for  $K$  can be computed from Chart 1.

Equation (3) shows that  $M_p$  is directly proportional to  $G$ ,  $K$ , and  $\phi'$ , whereas  $\phi'$  is indirectly proportional to  $GK$ , where  $GK$  is the *torsional rigidity* of the cross-section. Thus as  $K$  is increased,  $M_p$  increases. This means that  $M_p$  will offer a greater resistance to twisting action. The larger values for  $K$  appear for the larger (thicker) beam sections. For example,  $K = 68.80$  for a 36 WF 300 section, whereas,  $K = 0.195$  for a 5WF16 section.<sup>1</sup>

The influence of  $M_p$  is shown in Figs. 3a and 3b. In Fig. 3a, a solid circular cross-section is loaded at midspan with a torsional moment  $M = 1$ . At the ends,  $M_p = M_t = M/2$  and  $\phi = 0$ . Since a circular section cannot warp (warpfree),  $M_p$  is constant for each half of the beam. From Equation (1),  $M_t = M_p + 0$ , where  $M_s = 0$  and there is no warping. For circular sections, then,  $M_p = M_b$ , and  $M_p$  is exactly analogous to the shear diagram in plane bending. In Fig. 3b a WF section is loaded at midspan with a torsional moment  $M = 1$ . The noticeable difference between Fig. 3a and Fig. 3b is the curve for  $M_p$ . Since WF, I and C-sections are free

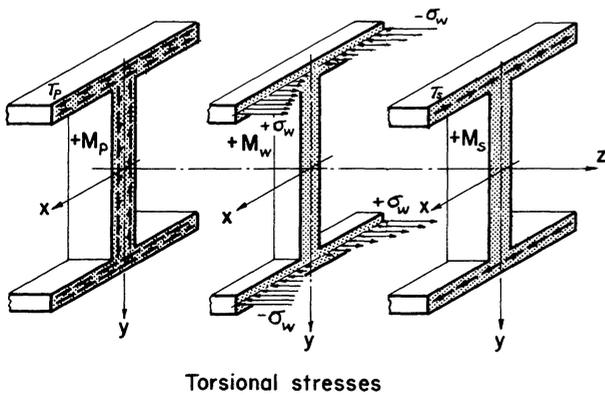


(a) Primary shear, circular section. ( $M_s = 0$ )



(b) Primary and secondary shear, WF-section

Figure 3



Torsional stresses

Figure 4

to warp if unrestrained,  $M_p$  no longer equals  $M_t$ . An increment of  $M_t$  is made up by the contribution of  $M_s$  as warping torsion. At midspan, where due to symmetry the cross-section remains planar,  $M_p = 0$  as in Fig. 3a.

**Influence of  $M_s$** —Figure 3b also shows the influence of  $M_s$ . At midspan where the cross-section is prevented from warping, the warping resistance is maximum.

Here  $M_s = M_t = M/2$ . Since the sum of  $M_p$  and  $M_s$  equals  $M_t$ , the curves for  $M_p$  and  $M_s$  are complementary.

As  $M_s$  expresses warping resistance, a distinction must be made between various profiles as to their warping characteristics. One way is to refer to a *warping resistance constant*, known as  $C_w$ . Approximately exact values for  $C_w$  are tabulated in Table 4 for all rolled sections commonly used. Other values for  $C_w$  can be computed for sections of built-up welded plates by using the equations in Chart 1 and 2. For rolled WF sections

$$C_w = \frac{I_y h^2}{2 \cdot 2} \text{ or } \frac{I_y h^2}{4} \quad (4)$$

Chart 1 shows the equations for  $C_w$  for several built-up cross-sections. For rolled sections,  $C_w$  is negligible for T and L-sections. WF, I and C-sections, on the other hand, are free to warp if unrestrained and warping effects must be considered. Sections made up of not more than *two* rectangular elements do not warp, because of the fact that the middle planes of each element pass through the shear center. For WF, I and C-sections, the middle planes of *every* element do not pass through the shear center (Chart 2).

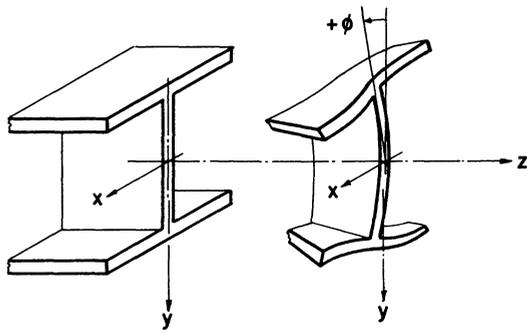
The warping resistance for moment  $M_s$  is determined from the equation

$$M_s = -EC_w \phi''' \quad (5)$$

where  $E$  is the modulus of elasticity and  $\phi'''$  is the third derivative of  $\phi$ . From this equation it is seen that the warping resistance constant  $C_w$  is directly proportional to the moment  $M_s$  and indirectly proportional to the angle of twist  $\phi$ . When  $C_w = 0$  as in circular sections,  $M_s = 0$ , as was shown in Fig. 3a. In Equation (5) the term  $EC_w$  is actually a measure of the *warping rigidity* of a cross-section. In Equation (3) the term  $GK$  is a measure of the *torsional rigidity* of a cross-section. Both of these terms, influenced by the geometry of the cross-section, are important to torsional behavior. The ratio of these rigidities appears in another torsional constant called  $\lambda$ , which is tabulated for all rolled sections in Table 4.  $\lambda$  is a constant that describes the rate of decrease of the warping stresses and is found from

$$\lambda = \sqrt{\frac{GK}{EC_w}} \text{ (in.}^{-1}\text{)} \quad (6)$$

**Torsional Stresses Resulting from  $M_p$  and  $M_s$** —The result of torsional moments  $M_p$  and  $M_s$  in a member is shearing stress. Figure 4 shows the cross-section of a WF profile under the influence of  $M_p$ ,  $M_w$  and  $M_s$ . The diagram for  $+M_p$  shows that shearing stresses are developed in the flanges and web as a result of  $M_p$ . The diagram for  $+M_s$  (warping) shows that the shearing stresses resulting from  $M_s$  appear only in the flanges.  $+M_p$  and  $+M_s$  follow sign convention when a positive



Positive angle of twist  $\phi$

Figure 5

angle of twist occurs as shown in Fig. 5. In terms of torsional shear magnitude, flange warping shear due to  $M_s$  is minor, compared to flange shear resulting from  $M_p$ ; hence,  $M_s$  is called a secondary resisting moment.

The torsional shearing stress  $\tau_p$  in the flanges or web of a WF, I or C-section due to  $M_p$  is determined from the equation

$$\tau_p = \frac{M_p t}{K} \quad (7)^*$$

where  $t$  is the thickness of flange or thickness of web.

Torsional shearing stresses in the flanges due to warping are found from

$$\tau_w = \frac{w_{\max} b M_s}{4 C_w} \quad (8)$$

where  $w_{\max}$ , shown in Chart 1, is the unit area of stress in the flange and  $b$  equals the width of the flange. For WF, I and C-sections  $w_{\max} = w_a$ . For C-sections  $w_a$  and  $w_b$  are required as shown in Chart 1. The values for  $w_a$  and  $w_b$  are tabulated in Table 4 for all rolled C-sections.

**Torsional Normal Stresses Resulting from  $M_w$** —Thus far the discussion has been centered around two effects of twisting, namely  $M_p$  and  $M_s$ , and the associated shearing stresses. Of all the torsional effects in WF, I and C-sections, the additive effect of torsional normal stresses  $\sigma_w$  is the major consideration. The normal stresses  $\sigma_w$  are a result of torsional flange bending (induced by warping restraint) and are shown in Fig. 4.  $+M_w$  is shown as the horizontal bending of the flanges producing normal stresses  $\sigma_w$  in the flanges. Torsional normal stresses  $\sigma_w$  must be added algebraically to the normal stresses  $\sigma_b$  produced by plane bending. Note

in Fig. 4 the signs for stress  $\sigma_w$  follow the sign convention previously mentioned.

Normal stresses in plane bending are determined in the well-known manner by dividing the bending moment by a factor called section modulus. In torsional behavior, Prof. Bornscheuer<sup>3, 4</sup> suggests that the factor  $S_w$  be used as a warping modulus, and  $M_w$  as a moment (expressed as lb-in.<sup>2</sup>),\* to describe flange bending. This flange bending due to torsion is expressed by the equation

$$M_w = -EC_w \phi'' \quad (9)$$

Note that this equation is identical to Equation (5), except that the second derivative of  $\phi$  is used here. The torsional normal stresses can be determined from

$$\sigma_w = \frac{M_w}{S_w} \quad (10)$$

In each of the Cases of Table 1, the two lower diagrams represent the curves for  $M_b$  and  $M_w$ . When plotted one above the other, the designer can quickly locate the ordinate to the curve which is maximum. Both the location and magnitude of the ordinate are given, and the additive effects of normal stresses can be quickly attained.

**Summary of Torsional and Shears**—In any WF, I or C-section under torsional loading, three internal moments occur, which are:

$$\begin{aligned} M_p &= GK\phi', \text{ which produces flange and web shear } \tau_p \\ M_w &= -EC_w \phi'', \text{ which produces flange bending and flange normal stresses } \sigma_w \\ M_s &= -EC_u \phi''', \text{ which produces warping shear } \tau_w \text{ in the flanges only.} \end{aligned}$$

From these equations it is seen that the only unknowns are the angle of twist  $\phi$  and its three derivatives. In designing for torsion,  $\phi$  is unknown; therefore, it has to be determined by the solution of the general differential equation.

However, the designer does not have to use this differential equation, since Table 1 has been set up for maximum torsional moments, which have been computed by determining the proper constants of integration and the location of maximum values. The simplified resulting equations are shown in each Case.

**$\lambda l$ -Curves and Their Significance\*\***—Figures 6a and 6b illustrate families of curves with varying parameters of  $\lambda l$ . One family represents the  $M_p$ -curves

\* Generally accepted expression for shearing stress in flanges or web. Reference 6 has, for the first time, more accurately determined this expression. In some cases, particularly C-sections, Equation (7) may be low by 20 per cent for values of shear in the flanges.

\* Bornscheuer uses the term "bimoment", since its dimension contains the second power of inches.

\*\* Reference 1 provides more extensive curve-plots for all rolled sections.

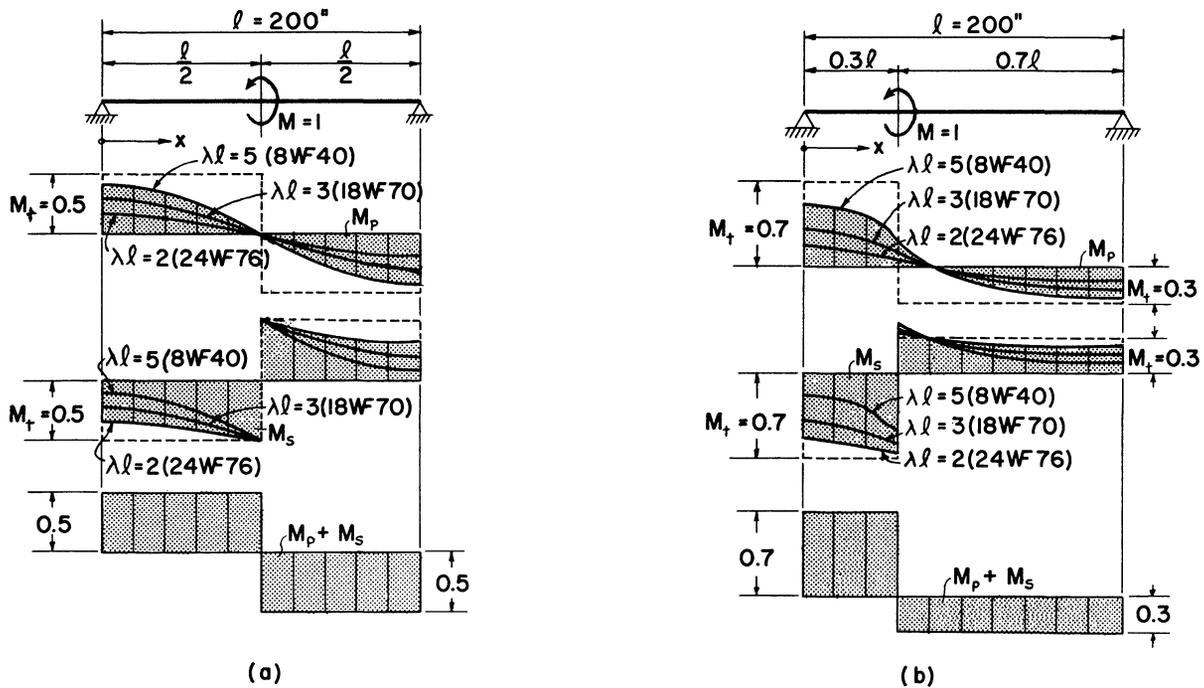


Figure 6

and the second family below it, the  $M_s$ -curves. The third diagram is the sum of the  $M_p$ - and  $M_s$ -curves and is a plot of Equation (1),  $M_t = M_p + M_s$ . As is true of any parameter,  $\lambda$  is fixed for each individual curve but differs from one curve to another in the same family.

Observation of Fig. 6a, with a unit torsional load at midspan, and Fig. 6b, with the same unit load at  $x = 0.3l$ , shows the complex nature of the curves involving hyperbolic functions. In Fig. 6a the family of  $M_p$ -curves ( $\lambda l = 5, 3$  and  $2$ ) each have the same point of inflection at midspan. At each end of the beam where  $x = 0$  and  $x = l$ , the ordinates at these points are maximum. As the parameter  $\lambda l$  increases, the  $y$ -intercept to the  $M_p$ -curves becomes larger. Graphically, this indicates that the influence of  $M_p$  is greater, and has more resistance to twisting than  $M_s$ . Examination of the family of  $M_s$ -curves shows that the point of maximum ordinate occurs at midspan where the curves pass through the point of inflection. As  $\lambda l$  increases, the  $y$ -intercept close to  $x = 0.5l$  becomes smaller and the influence of  $\lambda l$  is less noticeable.

By comparison, Fig. 6b shows the same family of curves, with different curvatures and a notable difference in the point of inflection for each  $\lambda l$ -curve for values of  $M_p$ . The point of inflection not only is not under the load as would be expected, but has different  $x$ -intercepts. The point of maximum ordinate to the  $M_p$ -curve is at  $x = 0$ . In the case of the  $M_s$ -curves the point of inflection occurs under the load at  $x = 0.3l$ . The maximum ordi-

nates to the curve are at the point of inflection. The higher values of  $\lambda l$  yield smaller  $y$ -intercepts and  $M_s$  contributes less in resisting the twisting of the beam.

The three curves of each diagram represent  $\lambda l = 2$ ,  $\lambda l = 3$  and  $\lambda l = 5$ . In order that their significance be meaningful in a practical sense, three beams have been selected from Table 4 whose  $\lambda$  values, when multiplied by the span length  $l$ , correspond to the  $\lambda l$  values plotted. The span length  $l$  is arbitrarily taken as 200 in. The beam sections and their  $\lambda$  values are as follows: 24 WF 76,  $\lambda = 0.0103$ ; 18 WF 70,  $\lambda = 0.01475$ ; 8 WF 40,  $\lambda = 0.0245$ .

Chart 3 gives the computed values for the ordinates to the  $M_p$ -,  $M_s$ - and  $M_t$ -curves in Fig. 6. It can be seen that the sum of the  $M_p$  and  $M_s$  ordinates all add to the sum of 0.50, which agrees with the diagram for  $M_t$  in Fig. 6a. The analogy between the shear diagram and the  $M_t$  diagram is strictly valid for this case of loading and end support. Therefore,  $M_t = 0.5M$ .

Figure 7 illustrates the family of  $M_w$ -curves located under the moment diagram for unit load at  $x = 0.3l$  producing plane bending. The parameters plotted are  $\lambda l = 5, 3$  and  $2$  as used in Fig. 6a and Fig. 6b. The maximum ordinate to the  $M_w$ -curve appears at  $x = 0.3l$  where each curve has a marked cusp. The maximum ordinate to the  $M_w$ -curve for symmetrical torsional loading also occurs under the point of loading and will correspond to the same location for the maximum ordinate to the  $M_p$ -curve.

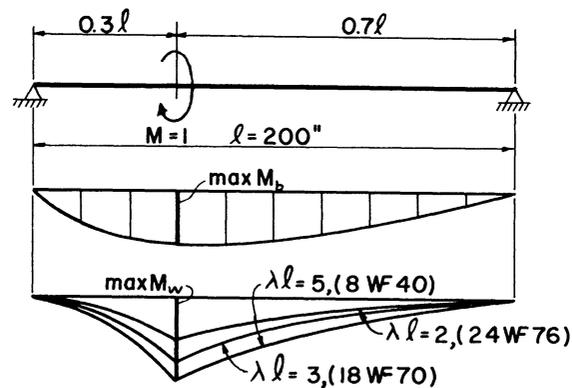
As the  $\lambda$  values increase, the  $y$ -intercept becomes greater. It can be seen from this figure that a greater warping resistance is required of a cross-section as  $\lambda$  increases. When heavier, thicker beam sections are selected, the ordinate to the  $M_w$ -curve becomes smaller, indicating that the geometry of the section has less tendency to warp and consequently the normal stresses developed will be of a lower magnitude.

The various Cases shown in Table 1 include only one curve for the parameter  $\lambda$ . It was not necessary to plot other  $\lambda$ -curves, since in practice the designer is primarily concerned with the location and magnitude of maximum ordinates to the curves. Equations to the right of the diagrams provide this information. After studying the diagrams provide this information. After studying the curves in Fig. 6a and Fig. 6b it will be evident that one curve of a family exhibits certain common characteristics compared to other curves in the family. All curves of the same family have the same concavity viz., concave upwards or concave downwards. All curves at any  $y$ -intercept have the same sign for curve slope  $m = \Delta y / \Delta x$  viz.,  $\Delta y$  increases or  $\Delta y$  decreases, indicating negative and positive slope respectively. Graphically this tells the designer the important condition as to whether the function under study is increasing or decreasing at any point along the beam span.

#### END RESTRAINTS

The condition of an end restraint is important in torsional analysis, as it is in plane bending. In building construction, and for *plane bending*, the AISC suggests three types of end restraint, namely: Type 1—fully-fixed beams, Type 2—simple connections, and Type 3—semi-rigid framing (partially restrained). In torsional analysis, only Types 1 and 2 will be considered (Fig. 8).

A Type 1 connection, in which the beam-end is fully welded around the flanges and web, offers only partial warping restraint and  $M_p \neq 0$ , as is found in many references. This restraint may range from 20 to 60 percent. In order to assume  $M_p = 0$  as shown in Fig. 8 and the diagrams in Table 1, the ends of the beam must be boxed-in. This can be simply taken care of by welding stiffener plates between the toes of the flanges. To be effective in the end zone of the beam, the length of these stiffeners along the longitudinal axis of the beam must be equal to or greater than the depth  $d$  of the beam. The designer should also take into consideration the torsional characteristics of the column in a beam-to-column connection. Where the rigid box-ended beam connects to a column with torsionally soft flanges it is advisable to provide column stiffeners between the flanges at the point of load application (see Design Example 1). In Type 2, which is considered as a typical web connection made up of two clip angles, the twisting at the connection is prevented and  $\phi = 0$ . However, it should be recognized that the  $L$  distance of the clip



Influence of  $\lambda l$  on  $M_w$

Figure 7

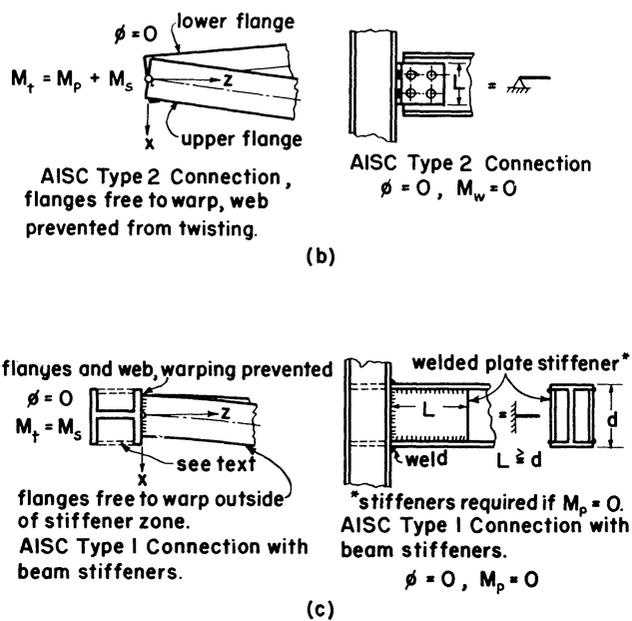
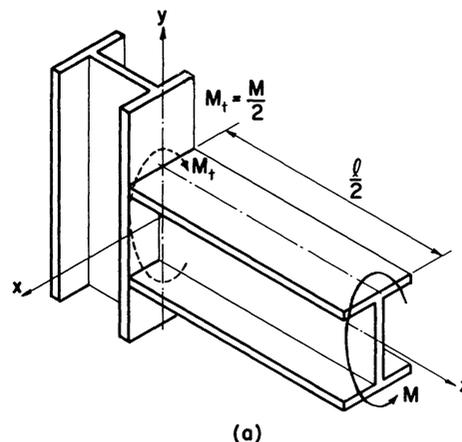


Figure 8

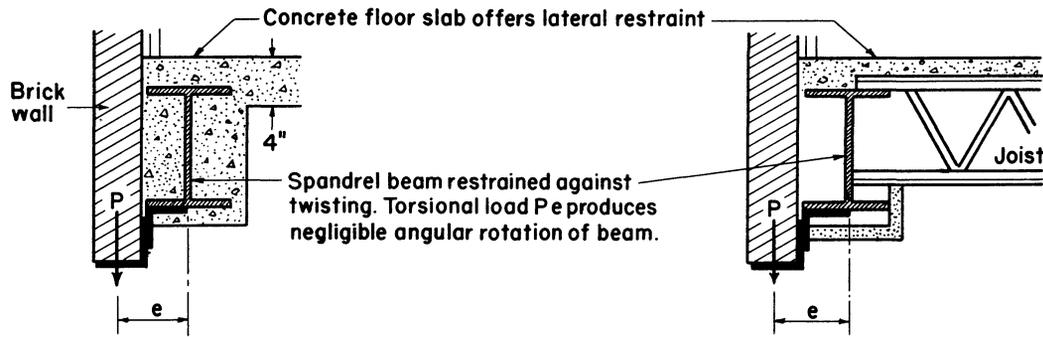


Fig. 9. Uniform torsional loading with no torsional stress effect

angles must extend over the major depth of the web in order that the assumption  $\phi = 0$  is reasonably valid.

What are the effects of end restraint on  $M_p$ ,  $M_w$  and  $M_s$ ? Figures 8b and 8c show Type 1 and 2 connections, where for Type 1,  $\phi = 0$  and  $M_p = 0$ ; for Type 2,  $\phi = 0$  and  $M_w = 0$ .

For torsional equilibrium,  $M_t = M_s$  as shown in Fig. 8c. In the Type 2 connection the beam does not twist at the connection. However, since the flanges can displace as shown in Fig. 8b, there is no restraint to flange bending and  $M_w = 0$ . The equilibrium of torsional moments is  $M_t = M_p + M_s$ . Actually,  $M_s$  would be zero at the ends if the beam were twisted by an equal and opposite moment at each end. This condition of pure torsion will rarely be found in structural practice.

#### PRACTICAL DESIGN CONSIDERATIONS

The occasions in actual practice where torsional problems arise are few in number, when considering building design. When such occasions occur, however, certain preliminary questions should be proposed. Will the torsional load produce significant twisting? Are there any restraining effects which will prevent twisting?

When an appreciable torsional moment is known to exist, the most satisfactory solution is to use a full length welded box girder. Usually with appreciable torsion, the box girder will take no more material than a heavy rolled section with welded end stiffeners.

In building design, most structural members are laterally restrained because of attachments to the structure along the length of the member. Rarely will the designer find a beam that is totally unrestrained viz., free to twist over its entire length. Hence, a good many cases involving torsional loading show that lateral restraints, existing in the form of attachments to the member, prevent twisting and torsional stresses can be ignored. This condition may be considered as torsion that is self-limiting. It is of no consequence when limited by the permissible end slope of the attaching members.

Based upon this evidence, beams that are a part of the floor assembly in buildings are usually restrained by a floor slab and, therefore, the torsion is simply self-limiting. Figure 9 illustrates this point where a typical spandrel beam is under a one-sided loading. The torsional load does not pass through the shear center. Normally, this loading would introduce torsional moments and subsequent torsional shearing and bending in the beam. In Fig. 9a the floor assembly consists of a concrete floor slab, usually not less than 4 in. in thickness, spanning not over 8 ft between intermediate beams. In order that the spandrel beam can twist under this eccentric loading, the entire floor assembly would have to rotate as well. Although the concrete slab in Fig. 9b is only 2½ in. thick, it also offers continuous and adequate lateral support against torsional twisting. Since twisting is prevented, it is safe to assume that normal stresses due to torsion can be ignored and no torsional analysis will be required.

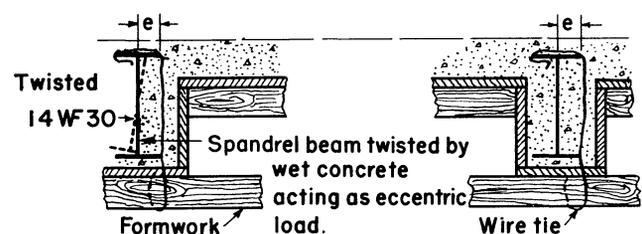


Fig. 10. Uniform torsional loading produces torsional stresses

**Beam Twists During Erection**—When significant twisting occurs in building construction, it will probably be during erection, before all the final loads are applied. During the erection of building structures, while temporarily unbraced, an unbalanced loading condition may produce excessive twisting. A few years ago, the writer investigated steel beams twisted by torsional loading as a result of improper field practice. The case involved spandrel and header beams in a school building,

which were twisted as much as 17 degrees. The twisting was due to unbalanced loading as shown in Fig. 10.

The formwork for the concrete slab was supported by wire ties wrapped around the top flange of the beams. All wire supports, however, were carried down on one side of the beam only. These wire ties supported the entire weight of the wet concrete. The one-sided support introduced an eccentricity equal to one-half of the flange width.

Since the beams were sized for plane bending only, they were torsionally weak. Had the wire ties been carried down on alternate sides of the beam, the unbalanced loading condition would never have occurred.

**Improving the Torsional Rigidity**—One means of improving the torsional rigidity of a rolled steel section is by the addition of new material, thus altering the  $K$ -value of the section—restricted, however, to that zone where the additional material occurs (Fig. 12). In other zones of the beam the effect is indirect. As an example, when a plate is welded between each flange at the toes, as in Fig. 11, the  $K$ -factor is increased considerably. In this instance  $K$  is increased to 3,246, approximately one hundred times the original value. The addition of these plates is analogous to the addition of cover plates to a beam under plane bending. It should be pointed out that the addition of these plates prevents the flanges from deforming; consequently warping effects become nil. The resulting closed box section is treated as being under primary shear only.

In passing, the engineer will recognize another familiar case of torsion and unsymmetrical bending resulting from biaxial bending, where a torsionally weak rolled section has to be reinforced. This is in the design of crane runway girders. It is common practice to assume that the bending moment caused by the horizontal loading is resisted by the upper flange. Accordingly, the top flange is reinforced.

#### GENERAL REMARKS

If torsional loading is known to produce significant twisting, and analysis of the torsional plus direct bending stresses shows the stresses are too high, the following solutions may be employed: (1) furnish a full length box girder section instead of a rolled section, (2) provide additional lateral supports, or braces, which will torsionally restrain the twist, (3) select a rolled section whose value of  $\lambda$  is of a lower value than the one which is highly stressed, and (4) consider the addition of welded plates between the flanges as just described (see Fig. 12). The improved torsional rigidity of steel beams encased in concrete for fire protection requirements should not be overlooked.

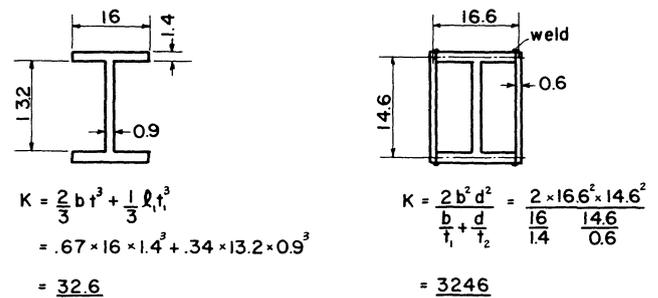


Fig. 11. Effect of stiffeners

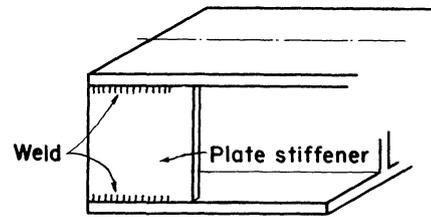


Fig. 12. Use of end stiffeners

#### DESIGN EXAMPLES

As stated previously, three types of torsional stresses are produced as a result of torsional loading on thin-walled open profiles such as a WF, I or C-section: (1) normal (bending) stresses in the flanges, critical at the toes, (2) primary shear stresses in the flanges and web, critical at the juncture of the web and flange, and (3) secondary warping shear stresses in the flanges, not critical and normally disregarded in stress analysis.

Of major concern in torsional analysis is the increase in bending stresses at the toes of the flanges which must be added to the direct bending stresses. Therefore, in any torsional design the first step is to determine the location of the maximum torsional moment  $M_w$ , and then determine the maximum normal stresses. Reference to Table 1 for the type of loading and nature of end restraint will give the location of the maximum ordinate to the  $M_w$ -curve. With a few minor exceptions, this location, as appearing in the diagrams, is applicable to all  $\lambda$ -curves of the same family. Where this is not true, an  $\bar{x}$ -distance is given for several  $\lambda$ -values. A linear interpolation can be made for intermediate values of  $\lambda$ .

The most frequently used loading condition is that of a concentrated torsional load  $M$  applied at some distance  $x$  from the left support, and a uniformly distributed load  $m$  applied over the entire span. For these conditions then, a "Short Method" is offered to the designer permitting him to evaluate the magnitude of

$M_w$  and  $\sigma_w$  immediately before continuing an extensive torsional analysis. The results obtained by this method are then compared with the permissible stresses for combined bending and torsion. If the results are close to the allowable stresses established, then a more exact method should be undertaken. The design examples illustrate both methods.

Under the "Short Method", each of the two loading conditions just mentioned are treated separately for two cases of end restraint. Table 2 covers Type 2 construction (simple framing connection) and Type 1 construction (all welded, fully fixed connection) is covered by Table 3.

When a more exacting analysis is required, the designer may refer to Table 1 using the appropriate Case for loading and end restraint. Equations are given for computing the maximum  $M_w$ ,  $M_p$  and  $M_s$ . For special conditions, an equation is given for  $M_w$  for any value of  $x$  along the span. With this information a complete  $\lambda$ -curve for  $M_w$  can be plotted. It will be found by the designer that under most circumstances the addition of torsional primary shear stresses is inconsequential.

Space does not permit the inclusion of the derivation of the general differential equation, nor the complete solution of this equation for numerous other conditions of loading and end support (see References 3, 13, 15).

### Example 1

*Given:* An 18W96 beam fixed at both ends is under a torsional load  $P$ , applied at the end of the bracket which is 71 in. long. (The eccentricity,  $e = 71$  in. The torsional moment  $M = 71 \times -400 = -28,400$  lb.-in.) (See Fig. 13.)

*Solution:*

From AISC Manual

$$I_x = 1674 \text{ in.}^4$$

$$S_x = 184.4 \text{ in.}^3$$

$$w = \text{D. L. beam}$$

From Table 4

$$w_{\max} = 50.90 \text{ in.}^2$$

$$I_w = 15,380 \text{ in.}^6$$

$$S_w = 302.1 \text{ in.}^4$$

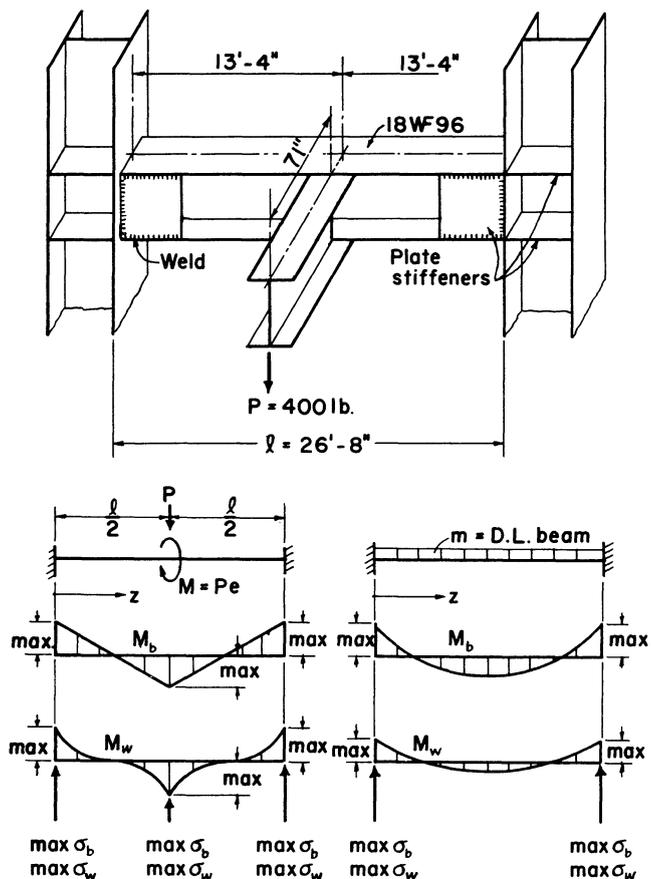
$$\lambda = 0.01201 \text{ in.}^{-1}$$

Normal stresses:

$$\sigma_b = \frac{M_b}{S}, \sigma_s = \frac{M_w}{S_w}$$

where

$M_b$  = plane bending and  $M_w$  = flange bending (torsion)



See TABLE 1 Case 10 & 5 and Short Method in TABLE 3.

Figure 13

1. From moment curves in diagram it is seen that the controlling condition will be when moments are combined at the end supports, for the determination of normal stresses.
2. Determine the moments and normal stresses due to plane bending:

Load  $P$ :

$$M_b = \frac{Pl}{8} = \frac{400 \times 320 \text{ in.}}{8} = -16,000 \text{ lb.-in.}$$

(max at center and ends)

$$\sigma_b = \frac{M}{S} = \frac{16,000}{184.4} = 86.7 \text{ psi}$$

Load  $w$ :

$$M_b = \frac{wl^2}{12} = \frac{96 \times (26.66)^2}{12} = -5,700 \text{ lb.-ft.}$$

(max at ends)

$$\sigma_b = \frac{M}{S} = \frac{5,700 \times 12}{184.4} = 371 \text{ psi}$$

3. Determine the moments and normal stresses due to warping torsion:

*Quick Method:* Use of influence lines in Table 3.

The following values are required:

Let  $\gamma = \lambda l = 0.01201 \times 320 = 3.84$ ,  $\eta = 0.370$ , where  $\eta$  is found by entering Table 3 with  $z/l = 160/320 = 0.5$ , proceed to curve where  $\gamma = 3.84$ . The desired value of  $\eta$  at the left is 0.370.

The moment  $M_w$ , due to flange bending, for  $z = 0$  is

$$M_w = \frac{1}{\lambda} \eta M = \frac{1}{0.01201} 0.370 \times (-28,400) = -876,000 \text{ lb.-in.}^2$$

The normal bending stress is

$$\sigma_s = \frac{M_w}{S_w} = \frac{876,000}{302.1} = 2,900 \text{ psi}$$

The normal stresses are as shown in Fig. 14.

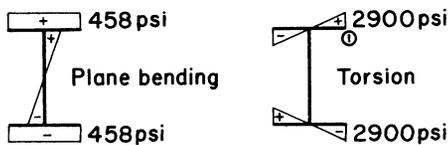


Figure 14

$$\text{At point } \textcircled{1}, \max \sigma = \max \sigma_b + \max \sigma_s = 458 + 2,900 = 3,358 \text{ psi}$$

*Conclusion:* Since  $\sigma_{\max}$ , due to combined bending and torsion, is well below any allowable stress limit it is unnecessary to proceed to a more extensive analysis.

### Example 2

*Given:* Assume the same conditions as Example 1 except for loading. Let  $m$  equal a uniformly distributed torsional load. Then  $m = 100 \times 7 = 700 \text{ lb.-in./ft}$  or  $58.4 \text{ lb.-in./in.}$  (See Fig. 15.)

*Solution:*

*Quick Method*

1. Determine warping moment  $M_w$  from influence lines Table 3.

Let  $\gamma = \lambda l = 0.01201 \times 320 = 3.84$  and  $1/\lambda^2 = 6,940$

Enter tabular values at right with  $\gamma = 3.84$  and find area  $A \cong -1.07462$ .

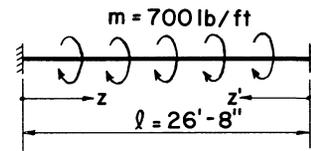
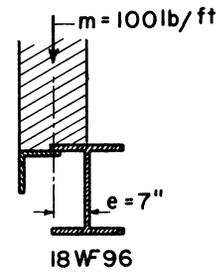


Figure 15

Warping moment  $M_w$  for  $z=0$  equals  $1/\lambda^2 Am$  where  $m = 58.4 \text{ lb.-in./in.}$

$$M_w = 6940 \times (-1.0746) \times 58.4 = -436,000 \text{ lb.-in.}$$

$$\max \sigma = \frac{-436,000}{302.1} = -1,450 \text{ psi}$$

This stress would then be added to any bending stresses resulting from plane bending.

*Exact Method*

Table I, Case 5

Warping moment  $M_w = -EC_w \phi''$

$$= \frac{m}{\lambda^2} \left( 1 - (1 - k) \frac{\sinh \lambda z + \sinh \lambda z'}{\sinh \lambda l} \right)$$

where

$$\lambda l = 3.84$$

$$\lambda z = 0.01201 \times 0 = 0$$

$$\lambda z' = \lambda l$$

$$k = 1 - \frac{\lambda(l/2)}{\tanh \lambda(l/2)} = 1 - \frac{1.92}{0.95792} = -1$$

$$M_w = 58.4 (6,940) \left( 1 - (1 + 1) \frac{0 + 22.5}{22.5} \right) = -406,000 \text{ lb.-in.}$$

$$\max \sigma = \frac{-406,000}{302.1} = -1,345 \text{ psi}$$

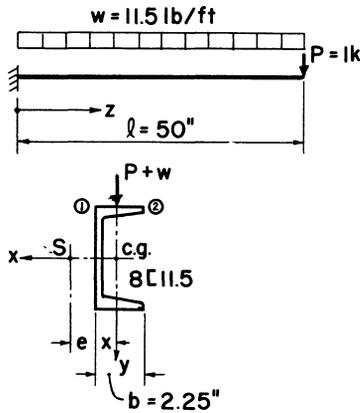


Figure 16

### Example 3

Given: Cantilever beam loaded at its free end with a concentrated load  $P = 1$  kip, acting through the c.g., point  $G$ . The dead load of beam will also be considered as acting through the c.g. (See Fig. 16.)

Solution:

From AISC Manual	From Table 4
$I_x = 32.3 \text{ in.}^4$	$\lambda = 0.05873 \text{ in.}^{-1}$
$S_x = 8.1 \text{ in.}^3$	$S_w = 2.423 \text{ in.}^4$
$x = 0.58 \text{ in.}$	$I_w = 12.84 \text{ in.}^5$
	$w_b = 2.555 \text{ in.}^2$
	$w_a = -5.3 \text{ in.}^2$
	$e = 0.5785 \text{ in.}$

Loads: D.L.  $w = 11.5 \text{ lb./ft} = 0.96 \text{ lb./in.}$

$P = 1 \text{ kip}$

Eccentricity:  $= e + x = 0.5785 \times 0.58 = 1.16 \text{ in.}$

Torsional moment due to D.L.:

$$-0.96 \times 1.16 = -1.11 \text{ lb.-in./in.} = -m$$

Torsional moment due to  $P$ :

$$-1000 \times 1.16 = -1,160 \text{ lb.-in./in.} = -M$$

The stresses at the fixed end are as follows:

Due to plane bending:

Bending moment due to  $w$ :

$$M_b = -\frac{0.96 \times 50^2}{2} = -1200 \text{ lb.-in.}$$

Bending moment due to  $P$ :

$$M_b = -1000 \times 50 = -50,000 \text{ lb.-in.}$$

Bending stresses due to  $w$ :

$$\sigma_b = \pm \frac{1,200}{8.1} = \pm 148 \text{ psi}$$

Bending stresses due to  $P$ :

$$\sigma_b = \pm \frac{50,000}{8.1} = \pm 6,170 \text{ psi}$$

Due to warping:

For warping calculations the following factors are needed:

$$\lambda^2 = 0.05873^2 = 0.00345$$

$$\lambda l = 0.05873 \times 50 = 2.94$$

From Table 2:

$$\sinh \lambda l = 9.431$$

$$\cosh \lambda l = 9.484$$

$$\tanh \lambda l = 0.9944$$

Warping moment due to  $w$ :

From Case ⑥ where  $z = 0$  and  $\lambda l \sinh \lambda z = 0$ :

$$\begin{aligned} M_w &= \frac{m}{\lambda^2} \left[ 1 + 0 - \frac{(1 + \lambda l \sinh \lambda l) \times 1}{\cosh \lambda l} \right] \\ &= \frac{-1.11}{0.00345} \left[ 1 - \frac{(1 + 2.94 \times 9.431)}{9.484} \right] \\ &= +654 \text{ lb.-in.}^2 \end{aligned}$$

Warping moment due to  $P$ :

From Case ⑦ where  $z' = l$  and  $\frac{\sinh \lambda l}{\cosh \lambda l} = \tanh \lambda l$ :

$$\begin{aligned} M_w &= \frac{M}{\lambda} \left( -\frac{\sinh \lambda l}{\cosh \lambda l} \right) \\ &= \frac{-1.11}{0.05837} \times (-0.9944) + 19,720 \text{ lb.-in.}^2 \end{aligned}$$

Warping stresses due to  $w$  at points ① and ②:

$$\sigma_2 = \frac{M_w}{S_w} \frac{+654}{-2.423} = -270 \text{ psi}$$

$$\sigma_1 = \frac{M_w}{I_w} \times w_1 \frac{+654}{12.84} \times 2.555 = +130 \text{ psi}$$

Warping stresses due to  $P$ :

$$\sigma_2 = \frac{+19720}{-2.423} = -8,140 \text{ psi}$$

$$\sigma_1 = \frac{+19720}{12.84} \times 2.555 = +3,920 \text{ psi}$$

The total longitudinal bending stresses in the upper flange at points ① and ② are:

$$\text{Pt. ①: } \sigma = +148 + 6,170 + 130 + 3,920 = +10,368 \text{ psi}$$

$$\text{Pt. ②: } \sigma = +148 + 6,170 - 270 - 8,140 = -2,092 \text{ psi}$$

The largest combination appears at Pt. ①.

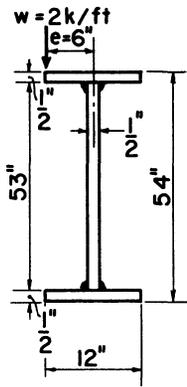


Figure 17

#### Example 4

*Given:* A beam built-up by welding three plates resembling an I-section is loaded with a uniformly distributed torsional moment  $m = 1000$  lb.-ft./ft over the entire span of 46 ft. The end connections are assumed to be bolted web clip angles, Type 2 construction. Determine if the additive torsional normal stresses are within permissible limits by the "Quick Method". ( $e = 6$  in.,  $w = 2,000$  lb./ft.) (See Fig. 17.)

*Solution:*

$$m = w_e = 2,000 \times 0.5 = 1,000 \text{ lb.-ft./ft (positive moment, see Fig. 5)}$$

Determine the torsional constants and properties from Chart 1:

$$C_w = \frac{1}{24} b^3 h^2 t = \frac{1}{24} 12^3 \times 53.5^2 \times 0.50 = 103,000 \text{ in.}^6$$

$$K = \frac{2}{3} b t^3 + \frac{1}{3} l_1 t_1^3 = \frac{2}{3} \times 12 \times 0.50^3 + \frac{1}{3} \times 53 \times 0.50^3 = 3.21 \text{ in.}^4$$

$$w_{\max} = \frac{b h}{4} = \frac{12 \times 53.5}{4} = 160.5 \text{ in.}^2$$

$$\lambda = 0.62 \sqrt{\frac{K}{C_w}} = 0.62 \sqrt{\frac{3.21}{103,000}} = 0.00341 \text{ in.}^{-1}$$

$$\lambda l = 0.00341 \times 552 = 1.88$$

$$S_w = \frac{C_w}{w_{\max}} = \frac{103,000}{160.5} = 642 \text{ in.}^4$$

Determine max. warping bimoment  $M_w$  at midspan by using Table 2, Quick Method. Since  $\lambda l = 1.88$ , a linear interpolation between  $\lambda l = 1.5$  and 2.0 gives 0.0893. Then  $\max. M_w = 0.0893 m l^2 = 0.0893 \times 1,000 \times 46^2 = 189,000$  lb.-ft.

$$\max \sigma_w = \frac{M_w}{S_w} = \frac{189,000}{642} = 3,540 \text{ psi}$$

Determine max. moment due to plane bending at midspan:

$$\max M_b = \frac{w l^2}{8} = \frac{2,000 \times 46^2}{8} = 528,000 \text{ lb.-ft.}$$

$$\max \sigma_b = \frac{M_b}{S} = \frac{528,000}{547} = 11,600 \text{ psi}$$

Combined normal stresses:  $\max \sigma_w + \max \sigma_b = 3,540 + 11,600 = 15,140$  psi (See Fig. 18.)

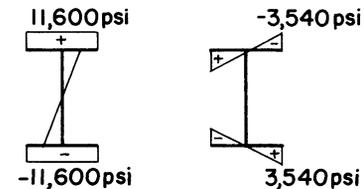


Figure 18

Check by long method using Table 1:

$$\max M_w = \frac{m}{\lambda^2} \left[ 1 - \frac{1}{\cosh \lambda l / 2} \right]$$

where  $\lambda l / 2 = 0.94$  and  $\cosh 0.94 = 1.475$

$$\max M_w = \frac{1000}{0.00341^2} \left[ 1 - \frac{1}{1.475} \right] = 27,800,000 \text{ lb.-in.}$$

By quick method:

$$27,800,000 \times 144 = 27,200,000 \text{ lb.-in.}$$

## NOMENCLATURE

<i>a</i>	Distance measured along the longitudinal axis of member (ft); distance from the shear center to the centerline of web in a channel (in.)
<i>b</i>	Width of a flange (in.); length of a rectangular element (in.); distance measured along the longitudinal axis of a member (ft)
<i>c</i>	Distance measured along the longitudinal axis of a member (ft)
<i>d</i>	Depth of a section (in.)
<i>e</i>	Eccentricity in a member; in a channel it is the distance from the shear center to the back of the channel web (in.)
<i>f</i>	Subscript indicating flange; symbol for function
<i>h</i>	Distance between centroids of flanges (in.)
<i>l</i>	Span of a beam (ft)
<i>l<sub>1</sub></i>	Distance between flanges (in.)
<i>m</i>	Uniformly distributed torsional load (lb.-ft/in.)
<i>t</i>	Thickness of a flange (in.)
<i>t<sub>1</sub></i>	Thickness of a web (in.)
<i>w<sub>a</sub>, w<sub>b</sub></i>	Value of unit warping at points a and b of a flange for WF, I and C-sections, (in. <sup>2</sup> )
<i>w<sub>max</sub></i>	Maximum value of unit warping, $w_{max} = w_a$ for WF and I sections (in. <sup>2</sup> )
<i>w<sub>1</sub>, w<sub>2</sub></i>	Value of unit warping at points 1 and 2 in the flanges of channel sections (in. <sup>2</sup> )
<i>x</i>	Coordinate; the distance in a channel from the back of the web to the <i>y</i> -axis (in.)
<i>y</i>	Coordinate; deflection in plane bending (in.)
<i>z</i>	Coordinate
<i>A</i>	Area of a section or rectangle (in. <sup>2</sup> ); area under influence line (in. <sup>2</sup> )
<i>A<sub>f</sub></i>	Area of flange (in. <sup>2</sup> )
<i>A<sub>w</sub></i>	Area of web (in. <sup>2</sup> )
<i>C</i>	Constant of integration
<i>C<sub>w</sub></i>	Warping resistance constant (in. <sup>6</sup> )
<i>E</i>	Modulus of elasticity (psi)
<i>G</i>	Shear modulus of elasticity (psi); center of gravity of a cross-section
<i>I</i>	Moment of inertia (in. <sup>4</sup> )
<i>K</i>	Torsional resistance constant; associated with St. Venant's torsion (in. <sup>4</sup> )
<i>L</i>	Length of stiffener plate (in.)
<i>M</i>	Applied concentrated torsional moment (lb.-ft)
<i>M<sub>b</sub></i>	Moment due to plane bending (lb.-ft)
<i>M<sub>p</sub></i>	Primary or pure torsional resisting moment (lb.-ft) associated with St. Venant's torsion
<i>M<sub>s</sub></i>	Secondary warping torsional resisting moment (lb.-ft); associated with warping torsion
<i>M<sub>t</sub></i>	Total torsional resisting moment at any given cross-section, where $M_t = M_p + M_s$ (lb.-ft)
<i>M<sub>w</sub></i>	Warping coefficient of loading and support condition; associated with warping forces and used to determine flange bending (lb.-in. <sup>2</sup> )

<i>P</i>	Concentrated load (lb.)
<i>S</i>	Shear center of a cross-section
<i>S<sub>w</sub></i>	Warping section modulus (in. <sup>4</sup> )
<i>S<sub>x, y</sub></i>	Section modulus about <i>x</i> or <i>y</i> axis (in. <sup>3</sup> )
<i>γ</i>	Parameter where $\gamma = \lambda l$ (dimensionless)
<i>θ</i>	Unit angle of twist (radians/in.)
<i>λ</i>	Torsional constant, $\lambda = \sqrt{GK/EC_w}$ (in. <sup>-1</sup> )
<i>ρ</i>	Radius of curvature (in.)
<i>σ</i>	Normal unit bending stress (psi)
<i>σ<sub>b</sub></i>	Normal stress associated with plane bending (psi)
<i>σ<sub>p</sub></i>	Normal stress associated with primary torsion (psi)
<i>σ<sub>s</sub></i>	Normal stress associated with warping torsion (psi)
<i>τ</i>	Unit shearing stress (psi)
<i>τ<sub>b</sub></i>	Shearing stress due to plane bending (psi)
<i>τ<sub>p</sub></i>	Shearing stress associated with St. Venant's primary torsion (psi)
<i>τ<sub>s</sub></i>	Shearing stress associated with secondary warping (psi)
<i>φ</i>	Total angle of twist (radians)
<i>EI<sub>x</sub>, EI<sub>y</sub></i>	Bending rigidity of a section (lb.-in. <sup>2</sup> )
<i>GK</i>	Torsional rigidity of a section (lb.-in. <sup>2</sup> )
<i>EC<sub>w</sub></i>	Warping rigidity of a section (lb.-in. <sup>4</sup> )

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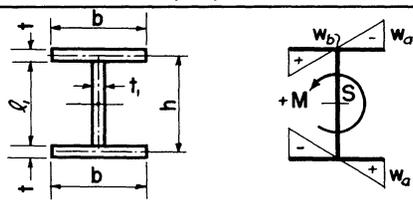
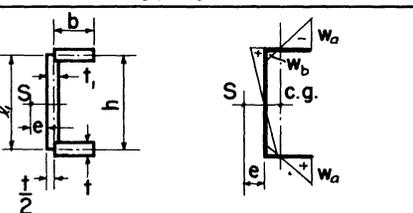
#### ACKNOWLEDGMENTS

The author expresses his sincere appreciation to Prof. F. W. Bornscheuer of the University of Stuttgart for his gratuitous offer in 1961 to compute, electronically, the torsional constants of American rolled sections appearing in Table 4. Other credits to Prof. F. W. Bornscheuer for valuable contributions included in this paper include: Table 3, influence lines for beams with welded end connections and the solutions for the differential equations from which Table 1 was prepared.

*Note: An extensive bibliography on the subject of torsion has been compiled by the author, and is available from AISC.*

#### CHARTS

CHART I Properties of Welded Shapes free to warp\*

	Doubly Symmetrical	Singly Symmetrical
		
$w_a = w_{max}$ (in <sup>2</sup> )	$\frac{bh}{4}$	$\frac{(A_1 + \frac{1}{3}A_2)b\ell_1}{4A_1 + \frac{2}{3}A_2}$
$w_b$ (in <sup>2</sup> )	—	$\frac{A_1 b \ell_1}{4A_1 + \frac{2}{3}A_2}$
$C_w$ (in <sup>6</sup> )	$\frac{1}{24} b^3 h^2 t$	$\frac{1}{12} b^3 \ell_1^2 t + \frac{3bt + 2\ell_1 t}{6bt + \ell_1 t_1}$
$K$ (in <sup>4</sup> )	$\frac{2}{3} b t^3 + \frac{1}{3} \ell_1 t_1^3$	$\frac{2}{3} b t^3 + \frac{1}{3} \ell_1 t_1^3$
$e$ (in)	—	$\frac{3 b A_1}{6 A_1 + A_2}$

Where  $A_1$  = flange area =  $bt$ ,  $A_2$  = web area =  $\ell_1 t_1$ , Total area =  $2A_1 + A_2$

$$\lambda = \sqrt{\frac{GK}{EC_w}} = 0.62 \sqrt{\frac{K}{C_w}} \text{ for steel}$$

\* Exact values for rolled sections are given in Table 4.

CHART 2 Warping Properties of Welded Shapes

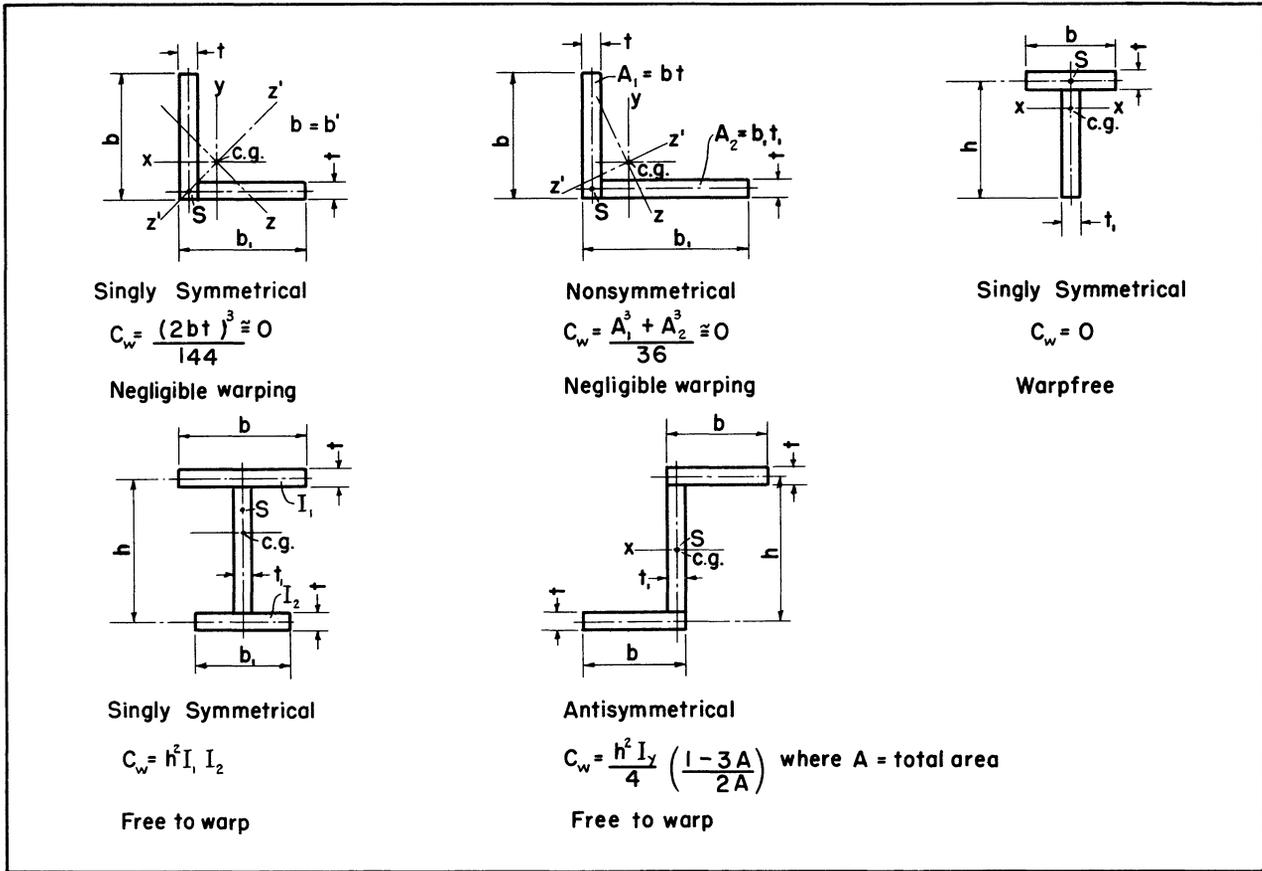
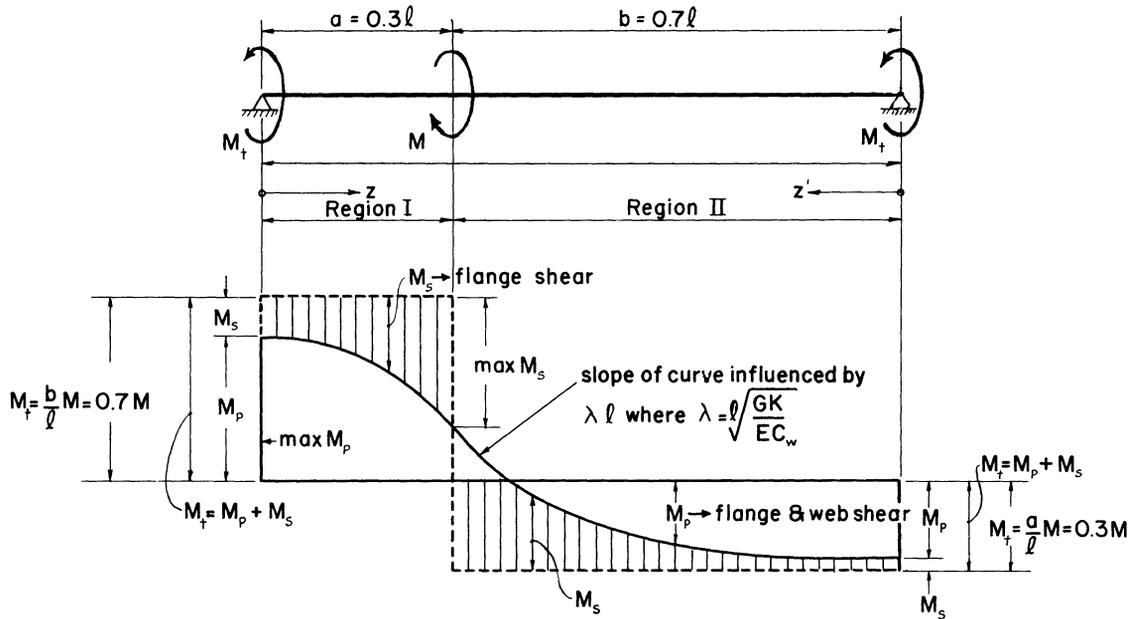


CHART 3

	x	$M_p$	$M_s$	$M_p + M_s$	$M_r = 0.5M$
$\lambda \rho = 2$ 24WF76	0.00	0.1760	0.3240	0.5000	0.50
	0.20	0.1497	0.3503	0.5000	0.50
	0.50	0	-0.5000 +0.5000	$\pm 0.5000$	$\pm 0.50$
	0.70	-0.1159	-0.3841	-0.5000	-0.50
	1.00	-0.1760	-0.3240	-0.5000	-0.50
$\lambda \rho = 3$ 18WF70	0.00	0.2875	0.2126	0.5000	0.50
	0.20	0.2480	0.2520	0.5000	0.50
	0.50	0	-0.5000 +0.5000	$\pm 0.5000$	$\pm 0.50$
	0.70	-0.1954	-0.3046	-0.5000	-0.50
	1.00	-0.2875	-0.2126	-0.5001	-0.50
$\lambda \rho = 5$ 8WF40	0.00	0.4185	0.0815	0.5000	0.50
	0.20	0.3742	0.1258	0.5000	0.50
	0.50	0	-0.5000 +0.5000	$\pm 0.5000$	$\pm 0.50$
	0.70	-0.3082	-0.1918	-0.5000	-0.50
	1.00	-0.4185	-0.0815	-0.5000	-0.50

	x	$M_p$	$M_s$	$M_p + M_s$	$M_r$
$\lambda \rho = 2$ 24WF76	0.00	0.1749	0.5251	0.7000	0.70
	0.20	0.1324	0.5676	0.7000	0.70
	0.30	0.0776	-0.3776 +0.8224	-0.3000 +0.7000	0.70
	0.80	-0.1102	-0.1898	-0.3000	-0.30
	1.00	-0.1245	-0.1755	-0.3000	-0.30
$\lambda \rho = 3$ 18WF70	0.00	0.2985	0.4015	0.7000	0.70
	0.20	0.2241	0.4759	0.7000	0.70
	0.30	0.1247	-0.4247 +0.5753	-0.3000 +0.7000	0.70
	0.80	-0.1785	-0.1215	-0.3000	-0.30
	1.00	-0.1975	-0.1025	-0.3000	-0.30
$\lambda \rho = 5$ 8WF40	0.00	0.4771	0.2229	0.7000	0.70
	0.20	0.3560	0.3440	0.7000	0.70
	0.30	0.1756	-0.4756 +0.5244	-0.3000 +0.7000	0.70
	0.80	-0.2557	-0.0443	-0.3000	-0.30
	1.00	-0.2713	-0.0287	-0.3000	-0.30

TABLE 1



Flange web shear  $T_p, T_s$  are determined from  $M_p$  and  $M_s$  respectively. Web shear is determined from  $M_p$  only. These torsional shears must be combined with the shears resulting from plane bending.

$M_t$  is determined, as above, from the symmetry of loading.

$M_p$  max., shown above, is determined from the following Tables.

$$M_s = M_t - M_p$$

Shears:

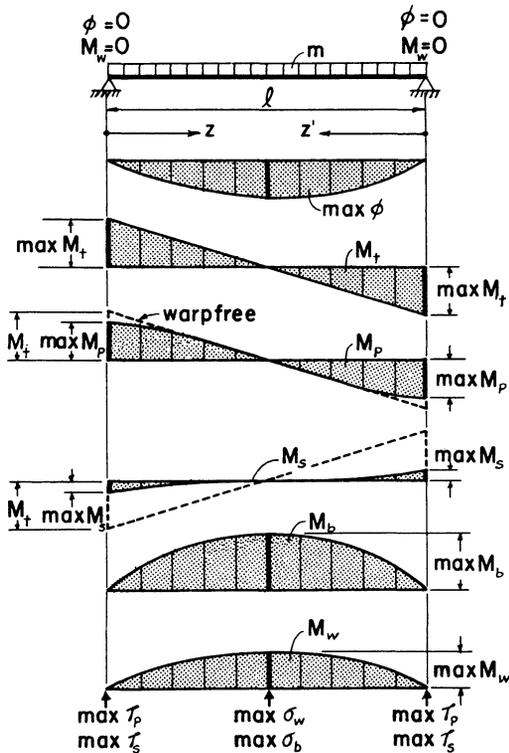
$$\text{Primary torsional shear stress, } \tau_p = G t \phi' = \frac{M_p t}{K}$$

where  $t$  is the thickness of the flange or web

$$\text{Secondary torsional shear stress, } \tau_s = \frac{w_{max} b}{4} \frac{M_s}{C_w}$$

where  $b$  is the width of flange

TABLE 1 (continued)



CASE 1

$$\max M_t = \frac{m l}{2}$$

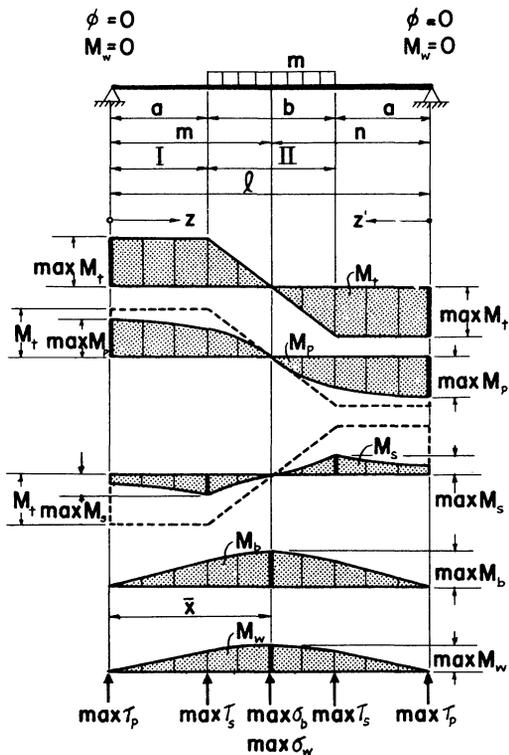
$$\max M_p = \frac{m}{\lambda} \left[ \frac{\lambda l}{2} - \tanh \frac{\lambda l}{2} \right], \text{ yields } \max \tau_p$$

$$\max M_s = \max M_t - \max M_p, \text{ yields } \max \tau_s$$

$$\max M_w = \frac{m}{\lambda^2} \left[ 1 - \frac{1}{\cosh \lambda \frac{l}{2}} \right], \text{ yields } \max \sigma_w$$

$M_w$  for any value of  $z$ :

$$M_w = \frac{m}{\lambda^2} \left[ 1 - \frac{\sinh \lambda z + \sinh \lambda z'}{\sinh \lambda l} \right]$$



CASE 2

$$\max M_t = \frac{m b}{2 l} [2 a + b]$$

$$\max M_p = \frac{m}{\lambda} \left[ \frac{\lambda b}{2} - \frac{\cosh \lambda (a + b) - \cosh \lambda a}{\sinh \lambda l} \right], \text{ yields } \max \tau_p$$

$$M_s = \max M_t - \max M_p, \text{ yields } \max \text{ combination of shear stresses}$$

$$\max M_w = \frac{m}{\lambda^2} \left[ 1 - \frac{\cosh \lambda a}{\cosh \lambda \frac{l}{2}} \right], \text{ yields } \max \sigma_w$$

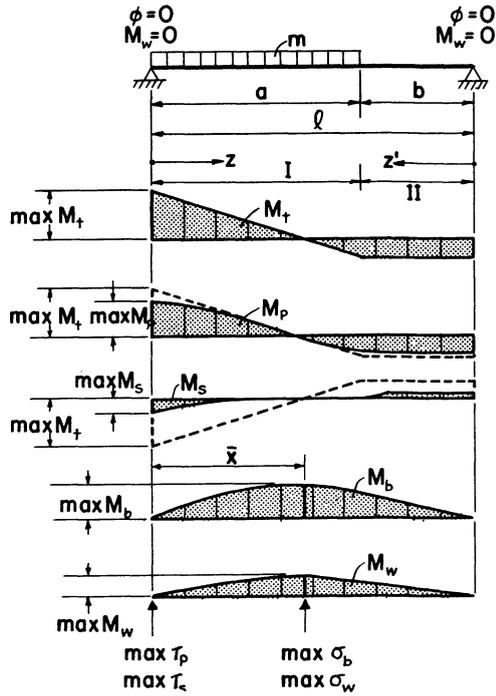
$M_w$  for any value of  $z$ :

$$\text{Region II, } M_w = \frac{m}{\lambda^2} \left[ 1 - \frac{\cosh \lambda a \cdot \sinh \lambda z' + \cosh \lambda a \cdot \sinh \lambda z}{\sinh \lambda l} \right]$$

$$\text{Region I, } M_w = \frac{m}{\lambda^2} \left[ \frac{\cosh \lambda (a + b) - \cosh \lambda a}{\sinh \lambda l} \right] \sinh \lambda z$$

For  $\max M_w$ , occurring in Region II, let  $z = \bar{x} + a + \frac{M_t}{m}$

TABLE 1 (continued)



CASE 3

$$\max M_t = \frac{m a}{2 \ell} (2 \ell - a)$$

$$\max M_p = \frac{m}{\lambda} \left[ \lambda \left( a - \frac{a^2}{2 \ell} \right) - \frac{\cosh \lambda \ell - \cosh \lambda b}{\sinh \lambda \ell} \right], \text{ yields } \max \tau_p$$

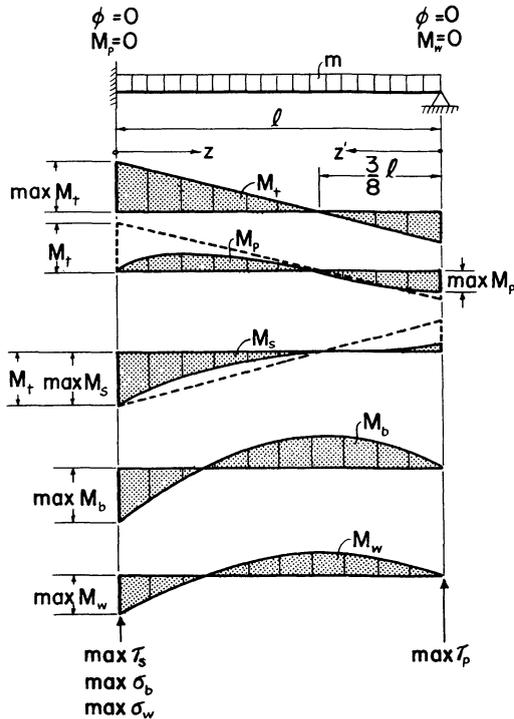
$$\max M_s = \max M_t - \max M_p, \text{ yields } \max \tau_s$$

$$\max M_w = A m \ell^2$$

$\lambda \ell$	A								
	$a=.10$ $\bar{x}=.09$	$a=.20$ $\bar{x}=.16$	$a=.30$ $\bar{x}=.23$	$a=.40$ $\bar{x}=.29$	$a=.50$ $\bar{x}=.34$	$a=.60$ $\bar{x}=.38$	$a=.70$ $\bar{x}=.43$	$a=.80$ $\bar{x}=.47$	$a=.90$ $\bar{x}=.49$
1.0	.004	.015	.030	.047	.065	.080	.094	.105	.111
2.0	.004	.014	.026	.039	.052	.064	.074	.082	.086
3.0	.004	.012	.022	.031	.040	.048	.055	.060	.063
4.0	.003	.010	.018	.025	.032	.036	.040	.043	.045
5.0	.003	.009	.015	.020	.024	.027	.030	.032	.033

$M_w$  for any value of  $z$ :

$$M_w = \frac{m}{\lambda^2} \left( 1 - \frac{\sinh \lambda z' + \cosh \lambda b \sinh \lambda z}{\sinh \lambda \ell} \right) \text{ Region I}$$



CASE 4

$$\max M_t = \max M_s$$

$$\max M_p = \frac{m}{\lambda} \left[ \frac{2k - (\lambda \ell)^2}{2 \lambda \ell} + \frac{\cosh \lambda \ell - (1+k)}{\sinh \lambda \ell} \right], \text{ yields } \max \tau_p$$

$$M_s = \frac{m}{\lambda} \left[ - \frac{\cosh \lambda \ell - (1+k)}{\sinh \lambda \ell} \right], \text{ yields } \max \tau_s$$

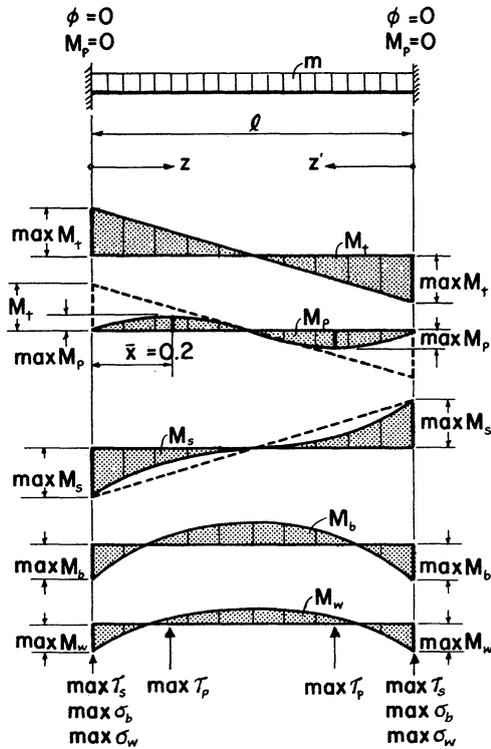
$$\max M_w = - \frac{m k}{\lambda^2}, \text{ yields } \max \sigma_w$$

$M_w$  for any value of  $z$ :

$$M_w = \frac{m}{\lambda^2} \left[ 1 - \frac{\sinh \lambda z + (1+k) \sinh \lambda z'}{\sinh \lambda \ell} \right]$$

$$\text{where } k = \lambda \ell \frac{\left( \lambda \frac{\ell}{2} - \tanh \lambda \frac{\ell}{2} \right) \tanh \lambda \ell}{\lambda \ell - \tanh \lambda \ell}$$

TABLE 1 (continued)



CASE 5

$$\max M_t = \frac{m \ell}{2}$$

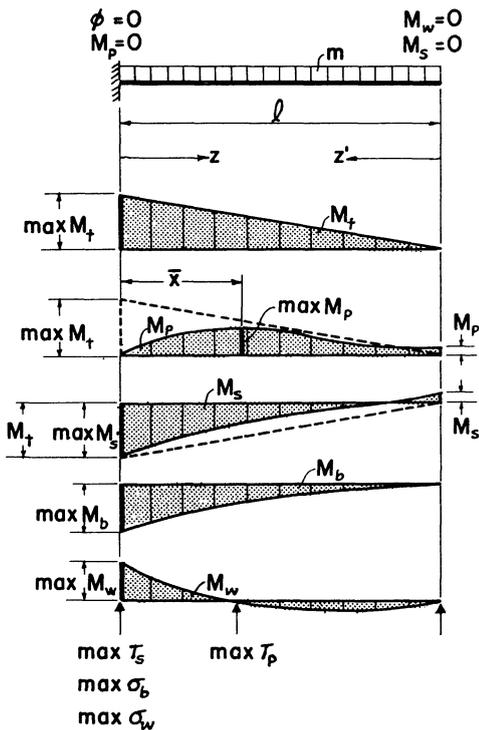
$$\max M_p = m \ell \left[ 0.30 + \frac{\cosh 0.2 \lambda \ell - \cosh 0.8 \lambda \ell}{4 \sinh^2 \frac{\lambda \ell}{2}} \right], \text{ yields } \max \tau_p$$

$$\max M_s = \max M_t, \text{ yields } \max \tau_s$$

$$\max M_w = \frac{m}{\lambda^2} \left[ 1 - \frac{\lambda \frac{\ell}{2}}{\tanh \lambda \frac{\ell}{2}} \right], \text{ yields } \max \sigma_w$$

$M_w$  for any value of  $z$  :

$$M_w = \frac{m}{\lambda^2} \left[ 1 - (1-k) \frac{\sinh \lambda z + \sinh \lambda z'}{\sinh \lambda \ell} \right] \text{ where } k = 1 - \frac{\lambda \frac{\ell}{2}}{\tanh \lambda \frac{\ell}{2}}$$



CASE 6

$$\max M_t = m \ell$$

$$\max M_p : \quad \max M_p = D m \ell$$

$\lambda \ell$	0.5	1.0	1.5	2.0	3.0	4.0	5.0
$\bar{x}$	1.0000	0.7719	0.6344	0.5371	0.4256	0.3667	0.3290
D	.0374	.1145	.1872	.2466	.3420	.4200	.4845

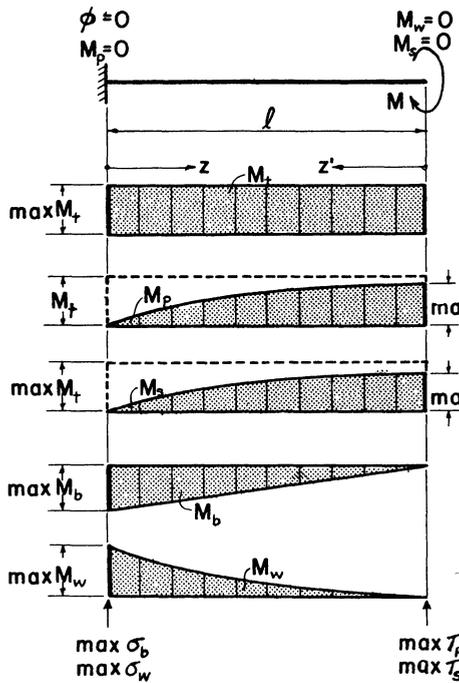
$$\max M_s = \max M_t - \max M_p, \text{ yields } \max \tau_s$$

$$\max M_w = \frac{m}{\lambda^2} \left[ \frac{1 - (1 + \lambda \ell \sinh \lambda \ell)}{\cosh \lambda \ell} \right], \text{ yields } \max \sigma_w$$

$M_w$  for any value of  $z$  :

$$M_w = \frac{m}{\lambda^2} \left[ 1 + \lambda \ell \sinh \lambda z - \frac{(1 + \lambda \ell \sinh \lambda \ell) \cosh \lambda z}{\cosh \lambda \ell} \right]$$

TABLE 1 (continued)



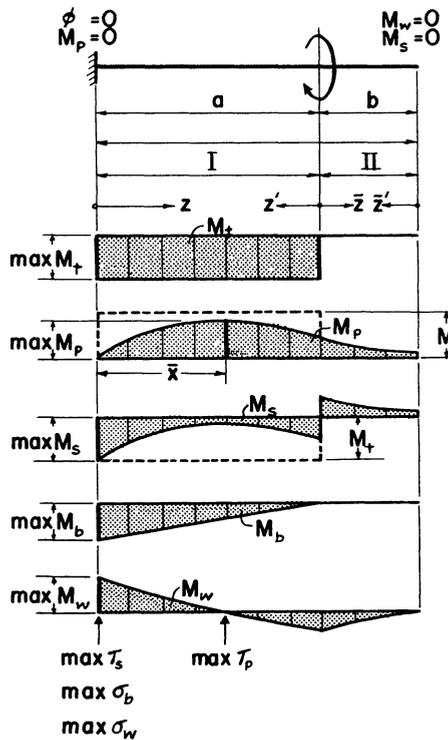
CASE 7

$\max M_t = M$   
 $\max M_p = M \left[ 1 - \frac{1}{\cosh \lambda \ell} \right]$ , yields  $\max \tau_p$   
 $\max M_s = \max M_t - \max M_p$ , yields  $\max \tau_s$

$\max M_w = -\frac{M}{\lambda} (\tanh \lambda \ell)$ , yields  $\max \sigma_w$

$M_w$  for any value of  $z$ :

$$M_w = \frac{M}{\lambda} \left[ -\frac{\sinh \lambda z'}{\cosh \lambda \ell} \right]$$



CASE 8

$\max M_t = M$

$\max M_p : M_p = DM, \bar{x} = a/\ell$   
 $a = 0.8 \quad a = 0.6 \quad a = 0.4 \quad a = 0.2$

$\lambda \ell$	1	2	3	4	5	1	2	3	4	5	1	2	3	4	5	1	2	3	4	5
$\bar{x}$	.75	.64	.56	.51	.48	.55	.46	.43	.43	.40	.36	.32	.29	.29	.27	.19	.17	.16	.16	.15
D	.22	.48	.64	.74	.82	.12	.31	.46	.58	.69	.06	.17	.29	.40	.50	.02	.06	.11	.17	.23

$\max M_s = \max M_t$ , yields  $\max \tau_s$

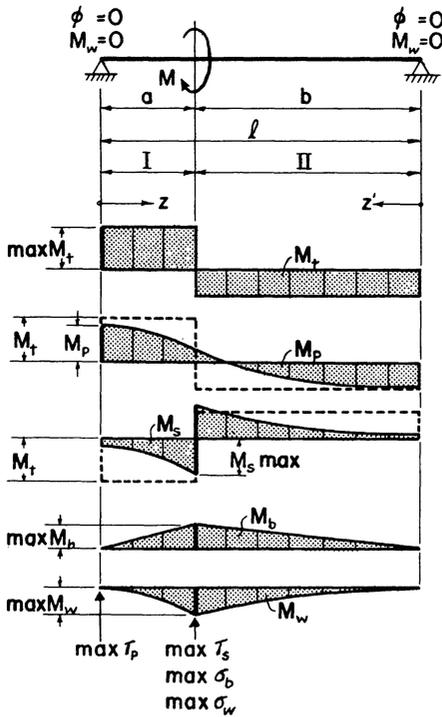
$\max M_w = \frac{M}{\lambda} \left[ -\sinh \lambda a + \tanh \lambda \ell (\cosh \lambda a - 1) \right]$ , yields  $\max \sigma_w$

$M_w$  for any value of  $z$ :

Region I,  $M_w = \frac{M}{\lambda} \frac{-\cosh \lambda \ell \sinh \lambda z' + (\cosh \lambda a - 1) \sinh \lambda \bar{z}'}{\cosh \lambda \ell}$

Region II,  $M_w = \frac{(\cosh \lambda a - 1) \sinh \lambda \bar{z}'}{\cosh \lambda \ell} \frac{M}{\lambda}$

TABLE 1 (continued)



CASE 9 For Quick Method see Table 3

$$\max M_t = \frac{Mb}{l}$$

$$\max M_p = M \left[ \frac{b}{l} - \frac{\sinh \lambda b}{\sinh \lambda l} \right], \text{ yields } \max \tau_p$$

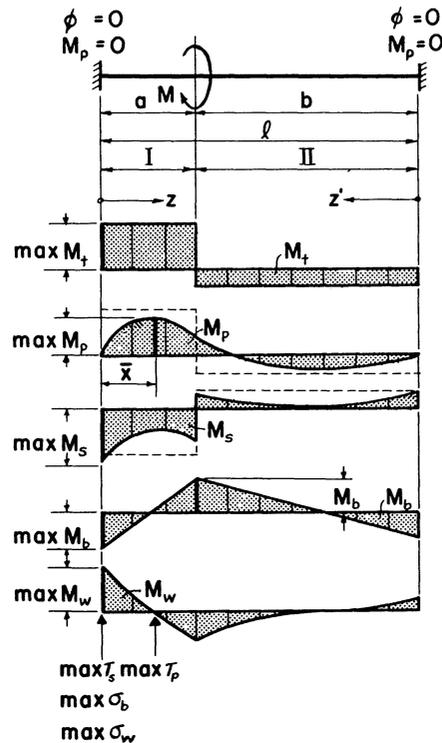
$$\max M_s = \max M_t - M_p, \text{ yields } \max \tau_s$$

$$\max M_w = \frac{M}{\lambda} \frac{\sinh \lambda b}{\sinh \lambda l} \sinh \lambda a, \text{ yields } \max \sigma_w$$

$M_w$  for any value of  $z$  :

$$\text{Region I, } M_w = \frac{M}{\lambda} \frac{\sinh \lambda b}{\sinh \lambda l} \sinh \lambda z$$

$$\text{Region II, } M_w = \frac{M}{\lambda} \frac{\sinh \lambda a}{\sinh \lambda l} \sinh \lambda z'$$



CASE 10 (when  $a < b$ ), See Table 3 for Short Method.

$$M_t = M_p + M_s$$

$$\max M_p = AM, \quad M_p = M \left[ \frac{\lambda b + k_2 - k_1}{\lambda l} - \frac{(\sinh \lambda b + k_2) \cosh \lambda z - k_1 \cosh \lambda z'}{\sinh \lambda l} \right]$$

$\lambda l$	A		
	$a=0.50l, \bar{x} \approx 0.25$	$a=0.30l, \bar{x} \approx 0.18$	$a=0.10l, \bar{x} \approx 0.10$
1.0	0.015	0.014	0.003
2.0	0.057	0.051	0.012
3.0	0.114	0.105	0.026
4.0	0.176	0.167	0.040
5.0	0.235	0.229	0.064

$$\max M_s = M \frac{(\sinh \lambda b + k_2) - k_1 \cosh \lambda l}{\sinh \lambda l}$$

$$\max M_w = \frac{M}{\lambda} k_1$$

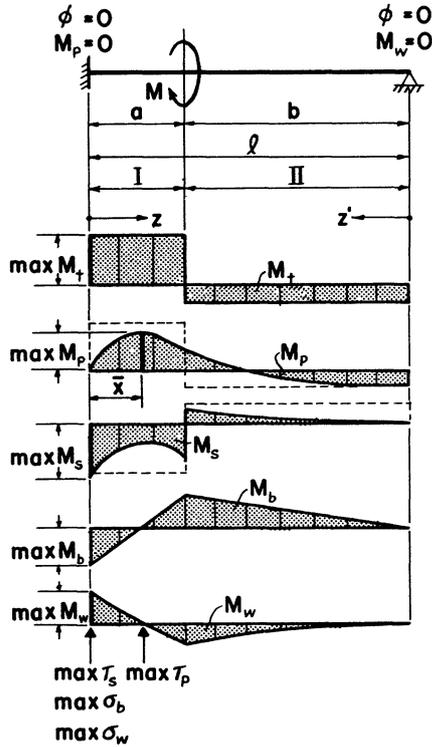
$M_w$  for any value of  $z$  :

$$\text{Region I, } M_w = \frac{M}{\lambda} \frac{(\sinh \lambda b + k_2) \sinh \lambda z + k_1 \sinh \lambda z'}{\sinh \lambda l}$$

$$\text{Region II, } M_w = \frac{M}{\lambda} \frac{k_1 \sinh \lambda z + (\sinh \lambda a + k_1) \sinh \lambda z'}{\sinh \lambda l}$$

$$k_{1,2} = \frac{\frac{\sinh \lambda a + \sinh \lambda b}{\sinh \lambda l} - 1}{2 \tanh \lambda \frac{l}{2}} \pm \frac{\left( \frac{a-b}{l} - \frac{\sinh \lambda a - \sinh \lambda b}{\sinh \lambda l} \right) \frac{l}{2} \tanh \lambda \frac{l}{2}}{l - 2 \tanh \lambda \frac{l}{2}}$$

TABLE 1 (continued)



CASE II

Region I  $M_p = M \frac{\lambda b - k}{\lambda \ell} - \frac{\sinh \lambda b \cosh \lambda z - k \cosh \lambda z'}{\sinh \lambda \ell}$

Region II  $M_p = M - \frac{\lambda a + k}{\lambda \ell} + \frac{\sinh \lambda a + k}{\sinh \lambda \ell} \cosh \lambda z'$

$\max M_p = A M$ , when  $a \geq b$   $\max M_p$  occurs at hinge support.  
when  $a < b$   $\max M_p$  occurs at fixed support.

$\lambda \ell$	Region I				Region II		
	$a=.10$ $\bar{x}=.08$	$a=.20$ $\bar{x}=.15$	$a=.30$ $\bar{x}=.19$	$a=.40$ $\bar{x}=.21$	$a=.50$ $\bar{x}=1.0$	$a=.70$ $\bar{x}=1.0$	$a=.90$ $\bar{x}=1.0$
1.0		.0112	.0176	.0242	-.0296	-.0350	-.0195
2.0	.0140	.0397	.0638	.0790	-.1025	-.1230	-.0702
3.0	.0296	.0809	.1258	.1525	-.1875	-.2299	-.1365
4.0	.0492	.1274	.1922	.2273	-.2602	-.3373	-.2058
5.0	.0717	.1781	.2573	.2945	-.3209	-.4138	-.2718

$\max M_s = \frac{\sinh \lambda b - k \cosh \lambda \ell}{\sinh \lambda \ell}$ , when  $a \leq b$ , occurs at fixed support.

$\max M_s = \frac{\sinh \lambda b \cosh \lambda a - k \cosh \lambda b}{\sinh \lambda \ell}$ , when  $a > b$ , occurs at load M.

$\max M_w = \frac{M}{\lambda} k$

$M_w$  for any value of  $z$  :

Region I,  $M_w = \frac{M}{\lambda} \frac{\sinh \lambda b \cdot \sinh \lambda z + k \sinh \lambda z'}{\sinh \lambda \ell}$

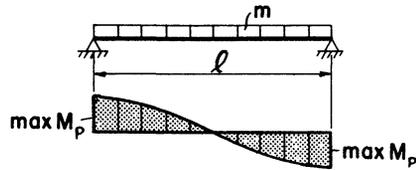
Region II,  $M_w = \frac{M}{\lambda} \frac{\sinh \lambda a + k}{\sinh \lambda \ell} \sinh \lambda z'$

$k = \frac{\lambda b \sinh \lambda \ell - \lambda \ell \sinh \lambda b}{\sinh \lambda \ell - \lambda \ell \cosh \lambda \ell}$

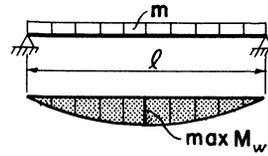
TABLE 2—SHORT METHOD

TYPE 2 CONSTRUCTION

Values for  $\max M_p$  and  $\max M_w$ , uniform torsional moment  $m$



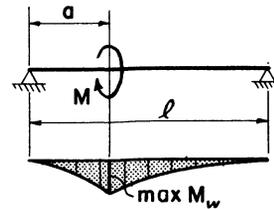
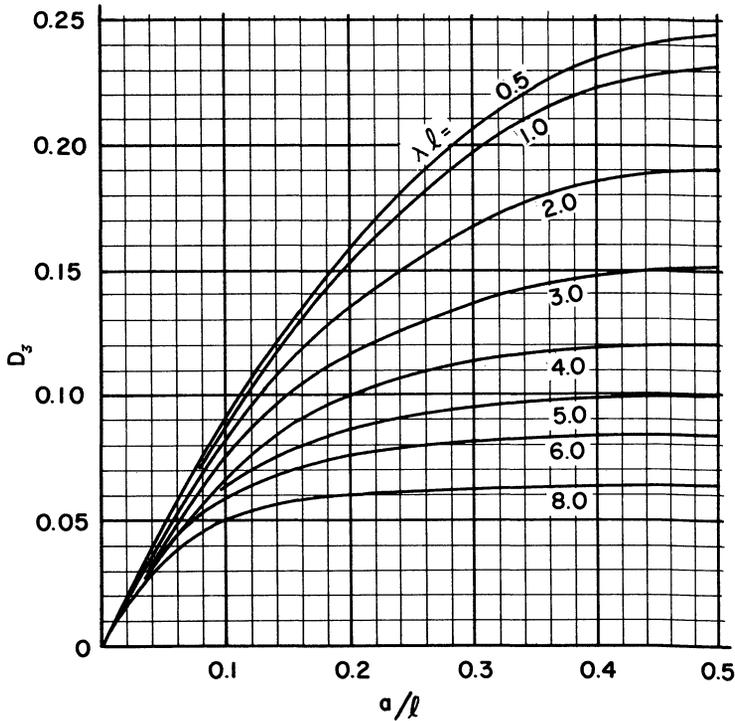
$$\max M_p = D_1 m l$$



$$\max M_w = D_2 m l^2$$

$\lambda l$	0.5	1.0	1.5	2.0	2.5	3	4	5	6
$D_1$	0.0102	0.0379	0.0765	0.1192	0.1607	0.1983	0.2590	0.3027	0.3342
$D_2$	0.1220	0.1132	0.1011	0.0880	0.0753	0.0639	0.0459	0.0338	0.0252

Values for  $\max M_w$ , concentrated torsional moment  $M$



$$\max M_w = D_3 M l$$

TABLE 3

INFLUENCE LINES FOR  $M_{w/z=0} f(\lambda l)$ , SHORT METHOD

$$M_{w/z=0} = \frac{1}{\lambda} \sum \eta M + \frac{1}{\lambda^2} A m$$

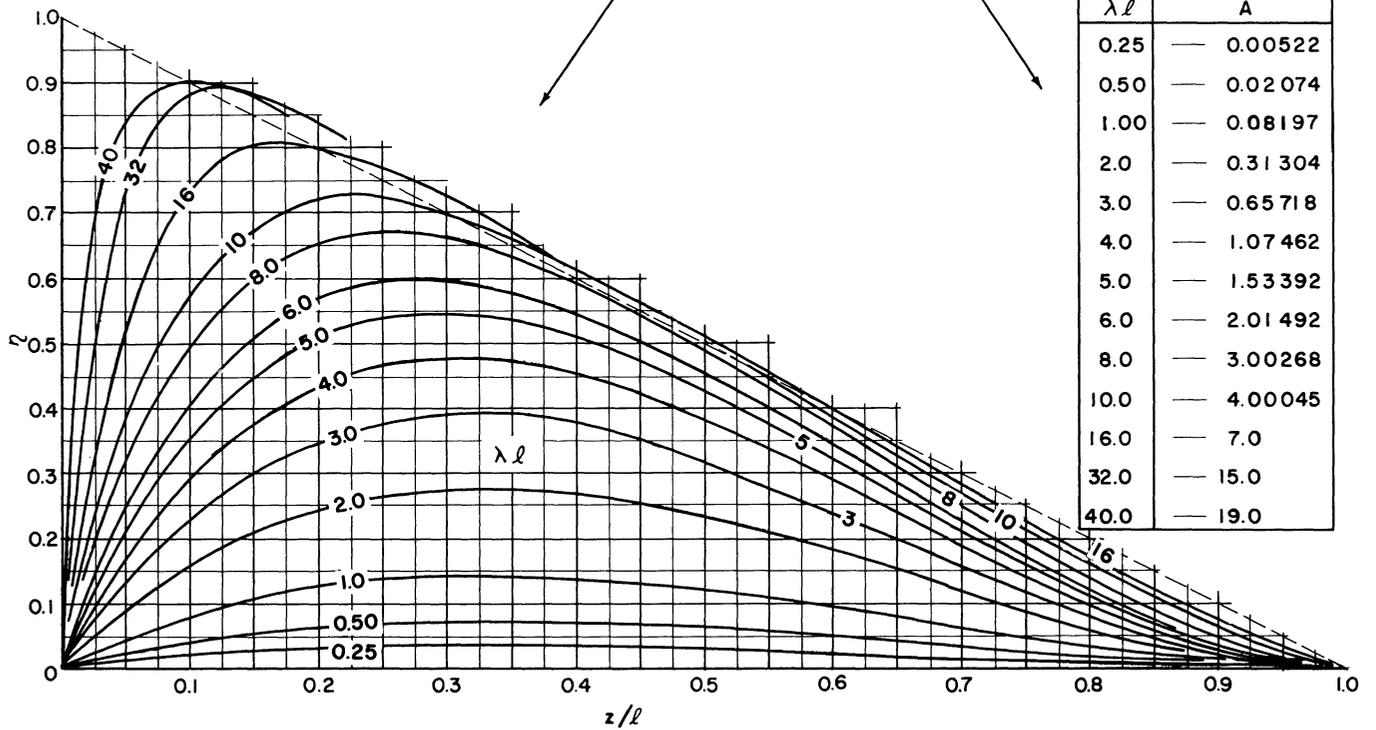
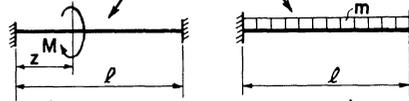


TABLE 4

Structural Section	$w_{max}$ in. <sup>2</sup>	$C_w$ in. <sup>6</sup>	$S_w$ in. <sup>4</sup>	$\lambda$ in. <sup>-1</sup>	Structural Section	$w_{max}$ in. <sup>2</sup>	$C_w$ in. <sup>6</sup>	$S_w$ in. <sup>4</sup>	$\lambda$ in. <sup>-1</sup>
36 WF 300	145.9	372,500	2,553	0.008284	18 WF 60	33.17	3,591	108.3	0.01556
280	144.9	340,700	2,351	0.007843	55	32.93	3,175	96.40	0.01451
260	144.0	306,000	2,125	0.007387	50	32.68	2,794	85.48	0.01343
245	143.3	281,800	1,967	0.007034	45	32.45	2,374	73.16	0.01237
230	142.6	258,400	1,812	0.006695	16 WF 96	44.53	12,240	274.9	0.01425
194	106.7	108,900	1,021	0.009050	88	44.18	10,820	245.0	0.01332
182	106.1	99,980	942.8	0.008594	78	33.15	5,167	155.9	0.01936
170	105.4	91,310	866.2	0.008133	71	32.82	4,548	138.6	0.01800
160	104.9	83,250	793.4	0.007756	64	32.48	3,957	121.8	0.01662
150	104.5	75,340	721.3	0.007379	16 WF 58	32.19	3,463	107.6	0.01542
135	103.8	61,710	594.6	0.006877	50	27.62	2,101	76.04	0.01700
33 WF 240	127.3	223,000	1,752	0.008106	45	27.38	1,825	66.68	0.01563
220	126.4	198,000	1,567	0.007567	40	27.12	1,576	58.12	0.01423
200	125.4	173,700	1,385	0.007011	36	26.96	1,302	48.29	0.01318
152	93.81	66,670	710.8	0.008613	14 WF 426	65.35	144,200	2,206	0.03001
141	93.29	59,420	637.0	0.008112	398	64.15	129,400	2,017	0.02875
130	92.78	51,730	557.5	0.007623	370	62.94	115,700	1,838	0.02745
118	92.22	43,380	470.4	0.007144	342	61.75	102,700	1,663	0.02606
30 WF 210	109.8	148,100	1,349	0.008788	314	60.50	90,460	1,495	0.02459
190	108.8	129,500	1,190	0.008114	287	59.35	79,270	1,336	0.02308
172	107.9	113,100	1,048	0.007502	264	58.34	70,470	1,208	0.02175
132	77.29	39,240	507.8	0.009972	246	57.55	63,830	1,109	0.02065
124	76.88	35,830	466.0	0.009480	237	57.16	60,590	1,060	0.02008
116	76.52	32,150	420.1	0.009010	228	56.77	57,530	1,013	0.01951
108	76.17	28,160	369.7	0.008570	219	56.36	54,400	965.1	0.01891
99	75.74	24,180	319.3	0.008104	211	56.04	51,700	922.6	0.01838
27 WF 177	92.01	87,620	952.4	0.009600	202	55.63	48,830	877.9	0.01777
160	91.17	76,720	841.5	0.008858	193	55.23	45,940	831.8	0.01714
145	90.44	67,600	747.5	0.008204	184	54.82	43,230	788.6	0.01652
114	66.33	25,670	387.0	0.010720	176	54.49	40,650	746.0	0.01593
102	65.73	22,050	335.5	0.009809	167	54.10	37,990	702.2	0.01527
94	65.34	19,470	297.9	0.009199	158	53.69	35,510	661.3	0.01461
84	64.89	16,050	247.3	0.008495	150	53.34	33,200	622.3	0.01399
24 WF 160	83.08	67,890	817.1	0.009813	142	53.04	30,900	582.6	0.01338
145	82.40	59,250	719.1	0.009057	320	61.48	88,130	1,433	0.02497
130	81.73	50,650	619.8	0.008293	136	50.44	26,570	526.9	0.01403
120	70.65	34,370	486.5	0.009792	127	50.03	24,460	489.0	0.01326
110	70.16	30,810	439.2	0.009140	119	49.67	22,600	455.0	0.01256
100	69.68	27,190	390.2	0.008474	111	49.33	20,710	419.8	0.01183
94	53.05	13,860	261.3	0.01227	103	48.96	18,940	386.8	0.01109
84	52.55	11,880	226.0	0.01121	95	48.62	17,150	352.7	0.01035
76	52.18	10,210	195.6	0.01034	87	48.26	15,490	320.9	0.009584
68	51.81	8,436	162.8	0.009525	84	40.28	10,120	251.2	0.01297
21 WF 142	66.86	39,640	592.9	0.01180	78	40.03	9,202	229.9	0.01216
127	66.14	34,410	520.2	0.01081	74	33.76	5,992	177.5	0.01576
112	65.44	29,090	444.5	0.009771	68	33.49	5,390	160.9	0.01467
96	45.65	11,030	241.7	0.01541	14 WF 61	33.17	4,716	142.2	0.01337
82	44.96	8,923	198.5	0.01364	53	26.77	2,534	94.67	0.01710
73	42.51	6,878	161.8	0.01330	48	26.54	2,236	84.25	0.01572
68	42.27	6,239	147.6	0.01259	43	26.30	1,948	74.07	0.01434
62	41.97	5,453	129.9	0.01170	38	23.05	1,231	53.42	0.01592
55	41.65	4,471	107.3	0.01074	34	22.86	1,065	46.60	0.01447
18 WF 114	51.74	19,360	374.3	0.01376	30	22.69	884.7	39.00	0.01304
105	51.32	17,340	337.8	0.01289	12 WF 190	40.05	23,520	587.3	0.02851
96	50.90	15,380	302.1	0.01201	161	38.78	18,640	580.8	0.02530
85	38.47	7,460	193.9	0.01718	133	37.54	14,360	382.5	0.02187
77	38.07	6,588	173.1	0.01592	120	37.00	12,440	336.1	0.02014
70	37.73	5,783	153.3	0.01475	106	36.37	10,630	292.4	0.01825
64	37.44	5,143	137.4	0.01375	99	36.05	9,727	269.8	0.01724
					92	35.75	8,864	248.0	0.01624

TABLE 4 (continued)

Structural Section	$w_{max}$ in. <sup>2</sup>	$C_w$ in. <sup>6</sup>	$S_w$ in. <sup>4</sup>	$\lambda$ in. <sup>-1</sup>	Structural Section	$w_{max}$ in. <sup>2</sup>	$C_w$ in. <sup>6</sup>	$S_w$ in. <sup>4</sup>	$\lambda$ in. <sup>-1</sup>
12 WF 85	35.42	8,059	227.5	0.01520	8 B 15	7.835	49.9	6.375	0.03321
79	35.16	7,329	208.4	0.01429	13	7.746	39.0	5.040	0.03059
72	34.85	6,542	187.7	0.01321	6 B 16	5.890	36.7	6.234	0.04882
65	34.54	5,784	167.5	0.01211	12	5.721	23.4	4.104	0.03948
58	28.91	3,577	123.7	0.01502	6 WF 25	8.989	149.3	16.62	0.03485
53	28.71	3,165	110.2	0.01389	20	8.776	113.3	12.92	0.02893
50	23.32	1,877	80.49	0.01915	15.5	8.597	79.5	9.250	0.02390
45	23.09	1,646	71.30	0.01755	5 WF 18.5	5.904	49.0	8.307	0.04840
40	22.85	1,437	62.88	0.01593	16	5.800	40.3	6.960	0.04315
36	19.20	805.2	41.94	0.02027	12 B 14	11.60	76.2	6.571	0.01960
31	18.96	663.6	34.99	0.01798	10 B 11.5	9.545	46.5	4.879	0.02107
27	18.79	549.5	29.25	0.01615	8 B 10	7.581	29.3	3.866	0.02463
10 WF 112	26.38	6,031	228.6	0.03116	6 B 8.5	5.551	14.9	2.685	0.03058
100	25.87	5,159	199.5	0.02863	5 M 18.9	5.729	45.6	7.963	0.05229
89	25.38	4,405	173.5	0.02616	4 M 13	3.571	12.4	3.487	0.07372
77	24.86	3,645	146.6	0.02334	24 I 120	46.07	10,640	231.0	0.02136
72	24.64	3,327	135.0	0.02208	105.9	45.08	10,010	222.0	0.01939
66	24.36	2,994	122.9	0.02058	100	41.90	6,108	145.8	0.02200
60	24.10	2,664	110.5	0.01904	90	41.19	5,823	141.4	0.01974
54	23.82	2,345	98.42	0.01742	79.9	40.45	5,544	137.0	0.01789
49	23.61	2,073	87.81	0.01602	20 I 95	34.35	4,339	126.3	0.02778
45	19.06	1,200	62.98	0.02192	85	33.65	4,094	121.7	0.02480
10 WF 39	18.80	994.0	52.88	0.01942	75	30.69	2,633	85.77	0.02602
33	18.55	791.0	42.65	0.01691	65.4	30.02	2,473	82.39	0.02289
29	14.09	356.0	25.26	0.02550	18 I 70	27.05	1,710	63.21	0.03146
25	13.90	292.4	21.04	0.02263	54.7	25.96	1,526	58.78	0.02398
21	13.74	220.1	16.02	0.01987	15 I 50	20.27	779.5	38.45	0.03274
8 WF 67	16.71	1,440	86.15	0.03699	42.9	19.77	727.1	36.78	0.02797
58	16.32	1,180	72.30	0.03311	12 I 50	15.53	479.7	30.89	0.04967
48	15.86	929.9	58.63	0.02857	40.8	14.89	426.2	28.63	0.03948
40	15.53	724.8	56.67	0.02453	35	14.54	313.7	21.57	0.03628
35	15.30	618.0	40.38	0.02198	31.8	14.32	300.6	20.99	0.03312
31	15.13	528.9	34.95	0.01984	10 I 35	11.75	176.6	15.03	0.05596
28	12.42	311.4	25.07	0.02575	25.4	11.08	150.1	13.56	0.03813
24	12.24	258.3	21.11	0.02266	8 I 23	7.899	58.6	7.426	0.06228
20	10.22	126.6	12.39	0.02813	18.4	7.575	52.2	6.901	0.04831
17	10.10	98.1	9.724	0.02515	7 I 20	6.377	32.6	5.113	0.07715
16 B 31	21.27	713.4	33.54	0.01610	15.3	6.046	28.1	4.658	0.05585
26	21.04	537.9	25.56	0.01412	6 I 17.25	5.028	17.1	3.402	0.09957
14 B 26	16.92	389.0	22.99	0.01905	12.5	4.696	14.1	3.019	0.06591
22	16.73	300.9	17.99	0.01680	5 I 14.75	3.837	8.3	2.164	0.1379
12 B 22	11.98	159.4	13.32	0.02676	10.0	3.506	6.4	1.847	0.08025
19	11.84	127.0	10.73	0.02390	4 I 9.5	2.591	2.9	1.134	0.1346
16.5	11.73	95.0	8.102	0.02196	7.7	2.465	2.5	1.039	0.1027
10 B 19	9.905	100.9	10.19	0.03003	3 I 7.5	1.719	1.0	0.5971	0.2093
17	9.815	82.1	8.370	0.02748	5.7	1.596	0.8	0.5242	0.1410
15	9.731	65.4	6.721	0.02556					

TABLE 4 (continued)

Structural Section	$e$ in.	$w_b$ in. <sup>2</sup>	$w_a$ in. <sup>2</sup>	$C_w$ in. <sup>6</sup>	$S_w$ in. <sup>4</sup>	$\lambda$ in. <sup>-1</sup>
18 [ 58.0	0.6617	8.753	-24.48	1,012	41.34	0.03307
51.9	0.7618	9.186	-23.62	931.2	39.42	0.02904
45.8	0.8707	9.697	-22.68	847.8	37.39	0.02555
42.7	0.9293	9.988	-22.17	804.8	36.30	0.02406
15 [ 50.0	0.4739	5.840	-17.50	391.5	22.37	0.04968
40.0	0.6418	6.339	-16.33	327.8	20.08	0.03935
33.9	0.7581	6.739	-15.51	287.8	18.55	0.03476
12 [ 30.0	0.4982	4.231	-11.96	116.6	9.752	0.05125
25.0	0.6120	4.529	-11.32	101.3	8.942	0.04270
20.7	0.7239	4.862	-10.69	87.35	8.169	0.03764
10 [ 30.0	0.2762	2.853	-9.564	60.49	6.325	0.08815
25.0	0.3897	3.043	-9.042	52.12	5.765	0.06956
20.0	0.5167	3.297	-8.454	43.69	5.169	0.05413
15.3	0.6575	3.634	-7.796	35.40	4.542	0.04468
9 [ 20.0	0.4122	2.661	-7.350	30.16	4.104	0.07154
15.0	0.5604	2.945	-6.732	23.99	3.564	0.05434
13.4	0.6183	3.074	-6.489	21.82	3.364	0.05081
8 [ 18.75	0.3406	2.161	-6.176	19.16	3.103	0.09204
13.75	0.4974	2.406	-5.600	14.85	2.652	0.06588
11.5	0.5785	2.555	-5.300	12.84	2.423	0.05873
7 [ 14.75	0.3523	1.810	-4.823	10.03	2.081	0.09926
12.25	0.4392	1.923	-4.546	8.58	1.888	0.08118
9.8	0.5370	2.074	-4.231	7.11	1.682	0.06923
6 [ 13.0	0.3001	1.421	-3.805	5.48	1.442	0.1296
10.5	0.3954	1.516	-3.548	4.54	1.282	0.1012
8.2	0.4984	1.645	-3.267	3.67	1.123	0.08400
5 [ 9.0	0.3492	1.157	-2.665	2.23	0.8378	0.1344
6.7	0.4613	1.261	-2.414	1.71	0.7111	0.1059
4 [ 7.25	0.3163	0.8480	-1.867	0.92	0.4978	0.1831
5.4	0.4194	0.9099	-1.688	0.69	0.4137	0.1409
3 [ 6.0	0.2445	0.5480	-1.241	0.33	0.2731	0.2991
5.0	0.3274	0.5938	-1.135	0.28	0.2472	0.2406
4.1	0.3931	0.6238	-1.052	0.23	0.2188	0.2057