A Cost Evaluation of Space Trusses of Large Span

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THIS STUDY is the first part of a larger economic study dealing with space trusses, domes and cable roofs. The domes and cable roofs considered are characterized by perimeter support around a circular or elliptical plan. This led to the search for a space truss with support at the perimeter.

The particular space truss treated in this work met the above requirement. The mechanics of the structural analysis have been discussed elsewhere by Kato, Takashani, Tsushima, and Hirata in a paper presented at the International Conference on Space Structures,¹ and are presented here in summary form. The methods of analysis and design used here are approximate and give rapid preliminary design solutions. As a consequence of this simplification more than a thousand different problems were solved.

More difficult than the analysis and design of the truss is the estimating of its cost. In endeavoring to solve this problem the author became indebted to two structural steel fabricators who spent considerable time examining the structural details and supplying him with fabrication and erection costs.

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STRUCTURAL ANALYSIS

Kato, et al.,¹ have discussed in detail a space truss called the Takenaka truss, composed of square pyramidal elements connected as shown in Fig. 1. In this truss scheme the top chord members lie on a grid diagonal with respect to the bottom chord. The joints in the two planes are connected by diagonals so that the resultant truss is composed of inverted square pyramidal elements arranged in a checkered pattern. Consequently, this truss has fewer members than other space trusses and it becomes necessary to investigate the stability of the system. Kato has proved that for stability one must, at least:

- 1. Provide simple vertical supports at the boundary.
- 2. Prevent horizontal displacements of the structure.
- 3. Add edge members along the boundary support lines to form triangles at the perimeter of the top plane. (These members prevent the relative rotation of the pyramidal elements.)

When these conditions are satisfied, the truss is stable and, moreover, is statically determinate for vertical loading.

By considering a decreasing grid spacing (center to center distance between the pyramidal elements very small compared to the truss sides) it is possible, in the limit, to reduce the difference equations developed by Kato to differential equations. Letting:

- q = transverse load per unit area, and
- M = bending moments per unit width in the x and y directions, which are equal, one obtains the equilibrium equation

$$\nabla^2 M = -q \tag{1}$$





Calling *h* the depth of the truss, L_d the length of the diagonal, and 2λ the lower grid spacing, the bar forces become:

$$N_T = -M\sqrt{2}(\lambda/h) \tag{2}$$

$$N_B = 2M(\lambda/h) \tag{3}$$

$$N_{D,N} = \pm \frac{dM}{dn} \left(\frac{2\lambda}{h}\right) (L_d) \tag{4}$$

where N_T , N_B and $N_{D,N}$ are the direct forces in the top chord, bottom chord and diagonal, respectively, and dM/dn is the derivative of M in the n (x or y) direction.

Neglecting the displacements due to shear, the differential equation for the vertical displacements w may be written:

$$\nabla^2 w = -2M/D \tag{5}$$

where

$$D = \frac{1}{2\lambda} EI = \text{truss rigidity per unit width}$$
$$I = \frac{\hbar^2 \sqrt{2} A_T A_B}{A_B + 2A_T}$$
$$A_T = \text{area of the top chord}$$
$$A_B = \text{area of the bottom chord}$$

Since the truss is simply supported around a rectangular perimeter a by b, one may take the first term of the Fourier series expansion of q:

$$q = \frac{16}{\pi^2} q_0 \sin \frac{\pi x}{a} \sin \frac{\pi y}{b}$$
(6)

and determine M and w approximately from Equations (1) and (5) as:

$$M = \frac{16}{\pi^4} \frac{q_0 a^2}{[1 + (a^2/b^2)]} \sin \frac{\pi x}{a} \sin \frac{\pi y}{b}$$
(7)

and

$$w = \frac{32}{\pi^6} \frac{q_0 a^4}{D[1 + (a^2/b^2)]} \sin \frac{\pi x}{a} \sin \frac{\pi y}{b}$$
(8)

With b = a and b = 2a the maximum bar forces and maximum displacement are:

1. With
$$b = a$$
:

$$N_B = 0.082q_0 a^2 \left(\frac{2\lambda}{h}\right) \tag{9}$$

$$N_T = -0.058q_0 a^2 \left(\frac{2\lambda}{h}\right) \tag{10}$$

$$N_D = 0.26q_0 a \left(\frac{2\lambda}{h} L_d\right) \tag{11}$$

$$w = 0.0083q_0(a^4/D) \tag{12}$$

Table 1. Comparison between Experimental Results and
Approximate and Exact Methods of Analysis

	Experimental	Theoretical			
	Experimental	Exacta	Approximate		
Displacement Top chord (#6) Bot chord (#10) Diagonal (#11)	30.0 mm -3,040 kg 6,200 kg 2,940 kg	34 mm -4,710 kg 7,020 kg 2,770 kg	32 mm -4,700 kg 7,420 kg 2,440 kg		

^a Tables 3 and 4 of paper by Kato, et al., Ref. 1.

2. With
$$b = 2a$$

$$N_B = 0.132q_0 a^2 \left(\frac{2\lambda}{h}\right) \tag{13}$$

$$N_T = -0.094 q_0 a^2 \left(\frac{2\lambda}{h}\right) \tag{14}$$

$$V_D = 0.416 q_0 a \left(\frac{2\lambda}{h} L_d\right) \tag{15}$$

$$w = 0.0213q_0(a^4/D) \tag{16}$$

In order to check the validity of these equations, Table 1 compares the results with those obtained by the more exact difference equations of Kato.

STRUCTURAL DESIGN

From Equations (4) and (7) one obtains the following expressions for the maximum diagonal bar forces in the x and y direction, respectively:

$$N_{D,X/\max} = \pm \frac{32}{\pi^3} \frac{q_0 a}{\left[1 + (a^2/b^2)\right]} \lambda \left(\frac{L_d}{h}\right)$$
(17)

$$N_{D,Y/\max} = N_{D,X/\max}(a/b) \tag{18}$$

The average diagonal bar force may be determined by a summation which, in the limit as 2λ goes to zero, reduces to an integration:

$$N_{D,X/avg} = (4/\pi^2) N_{D,X/max}$$
 (19)

$$N_{D,Y/avg} = (4/\pi^2) N_{D,Y/max}$$
(20)

Letting the allowable tensile stress F_a equal $0.6F_y$, the maximum and average areas for the tension diagonals are:

$$A_{X,T/avg} = \frac{214}{\pi^5} \alpha \frac{q}{F_y} \lambda^2 \frac{a}{h} \frac{\left[1 + (h^2/\lambda^2)\right]^{1/2}}{\left[1 + (a^2/b^2)\right]} \quad (21)$$

$$A_{Y,T/avg} = A_{X,T/avg}(a/b)$$
⁽²²⁾

$$A_{X,T/\max} = A_{X,T/\arg} \pi^2/4$$
 (23)

$$A_{Y,T/\max} = A_{Y,T/\alpha vg} \pi^2 / 4$$
 (24)

Compression Diagonal				Tension Diagonal						
Member Designation	A (i	rea n.²)	$\beta = \frac{A}{r_y^2}$		Mem Design	ber ation	Area (in.²)		$\beta = \frac{A}{r_y^2}$	
4₩F13	3	. 82	3 .90		3 I 5	i.7 7	1.64		5.8	
6 WF 25	7	. 37	3.20		41/	. /	2.21		0.2	
8 W F48	14	. 11	3.27				Avg $\beta = 6.0$			
8 WF 67	19	.70	4.39			Botto	om Chord			
10 W F89	26	.19	3.79		Mem	her	Area		Δ	
10 WF 112	32	. 92	4.62		Design	ation	(in^2)		$\beta = \frac{\pi}{r^2}$	
12 WF 161	47	. 38	4.62		Designation		(111.)		/ y	
14 \v =237	69	. 69	4.15		$\begin{array}{c} 2 - 2 \frac{1}{2} \times 2 \times \frac{3}{16} \\ 2 - 5 \times 3 \times \frac{3}{3} \\ \end{array}$		1.62 5.72		2.61	
		Ave	$\beta = 3.99$		$2 - 7 \times 4 \times$	1/2	7.96		2.96	
	$\operatorname{Avg} p = 3.35$		5 6 0.00		$2-8 \times 4 \times$	$\frac{1}{2}$ \angle	11.50		4.50	
								Av	$\log \beta = 3.34$	
				Ĵ	Гор Chord					
Member Area Designation A (in.		Area (in.²)		$\gamma = \frac{r_x}{d_1} \qquad \qquad k = \frac{y_1}{d_1} \qquad \qquad \eta =$		$\eta = \frac{d_1}{A^{\frac{1}{2}}}$				
ST 4-8	ST 4- 8 5		2 50		0.283	0.210)		2 53	
ST 5-14.5		4	4.27		0.274	0.206			2,50	
ST 8—32		9	9.40		0.284	0.216			2.60	
ST 9-52.5		15.43		1	0.270	0.199			2.35	
SI 10-/1 ST 13		20	20.88		0.2/1 0.202		2		2.36	
ST 16—110		32	20.05 32.37		0.282		0.213		2.60	
ST 18-130		38	3.28	•	0.282 0.2		5	2.61		
ST 18—150 44.09			0.285 0.224 2		2.70					
	Avg $\gamma = 0.279$ Avg $k = 0.212$ Avg $\eta = 2.54$									

Table 2. Cross-Sectional Area Properties

For definition of r_x , d_1 and y_1 see Ref. 65.

The ratio

$$\alpha = \frac{\text{actual bar area}}{\text{theoretical bar area}}$$
(25)

is introduced in Equation (21) to allow for the fact that the actual cross-sectional area is usually greater than the theoretical one required. For rolled shapes $\alpha = 1.1$. The average areas given above do not reflect the fact that the minimum area is limited by the maximum allowable slenderness ratio L/r. A further adjustment in this value is made in Equations (51) through (54).

In the design of the compression diagonals it is necessary to consider buckling. An examination of a sample of wide-flange shapes which may be used as compression diagonals reveals that:

$$r_y^2 \cong A/\beta \tag{26}$$

with β approximately constant (Table 2). Furthermore, for this problem it can be shown that for the compression diagonals:

$$\frac{KL_d}{r} > C_c = \sqrt{\frac{2\pi^2 E}{F_u}} \tag{27}$$

with K = 0.7. It follows that the allowable stress is given by:

$$F_a = \frac{1}{2K^2} \frac{\pi^2 E}{(L_d/r_y)^2}$$
(28)

Using the maximum and average diagonal forces previously considered, one obtains the corresponding areas:

$$A_{X,C/avg} = \frac{16\sqrt{2}}{\pi^3} \frac{K\lambda^2}{\sqrt{\pi}} \left(1 + \frac{h^2}{\lambda^2}\right) \times \left[\alpha\beta_1 \frac{q_0}{E} \frac{a}{h} \frac{1}{[1 + (h^2/\lambda^2)]^{1/2}} \frac{1}{[1 + (a^2/b^2)]}\right]^{1/2} \quad (29)$$
$$A_{Y,C/avg} = A_{X,C/avg} (a/b)^{1/2} \quad (30)$$

$$A_{X,C/\max} = \frac{\pi}{2} A_{X,C/\max}$$
(31)

$$A_{Y,C/\max} = \frac{\pi}{2} A_{Y,C/avg}$$
(32)

The maximum and average chord forces are derived in an analogous manner, with the result that for the compression (top) chords:

$$N_{T,C/avg} = \frac{128}{\sqrt{2}\pi^6} \frac{\lambda}{h} \frac{q_0 a^2}{\left[1 + (a^2/b^2)\right]}$$
(33)

$$N_{T,C/\max} = \frac{\pi^2}{4} N_{C,T/\text{avg}}$$
(34)

For the tension (bottom) chords:

$$N_{B,T/avg} = \sqrt{2}N_{T,C/avg} \tag{35}$$

$$N_{B,T/\max} = \sqrt{2}N_{T,C/\max} \tag{36}$$

For the bottom chords, which are in tension, the allowable stress is $F_a = 0.6F_y$ and the areas are:

$$A_{B,T/\text{avg}} = \frac{\alpha}{\epsilon} \frac{213}{\pi^6} \frac{\lambda}{h} \frac{a^2}{\left[1 + (a^2/b^2)\right]} \frac{q}{F_y}$$
(37)

$$A_{B,T/\max} = \frac{\pi^2}{4} A_{C,T/avg}$$
 (38)

where

$$\epsilon = \frac{\text{effective cross-sectional area}}{\text{net cross-sectional area}}$$
(39)

= 0.90 for bolted connections

For the top chord, besides the compressive force, it is also necessary to consider the bending moments that result from the fact that the metal deck spans between the top chord members.* It is assumed that this deck spans in one direction for the pyramidal elements and in the orthogonal direction for the bays in between. This results in a uniform reaction $q_f \times L_c/2$ from the metal deck where

- q_f = the uniform load consisting of the full live load plus the weight of the deck and roofing
- $L_c =$ length of the top chord bars

It should be noted that the uniform load q carried by the space truss is the full dead load, including the weight of the truss, plus the live load which may be reduced by a live load reduction factor RF:

$$RF = \frac{\text{uniform live load carried by the truss}}{\text{uniform live load carried by the top chord}}$$
(40)

Considering a tee section, it is necessary to define the following constants based on cross-sectional area properties (Table 2):

$$\gamma = r_x/d_1 = 0.279 \tag{41}$$

$$k = y_1/d_1 = 0.212 \tag{42}$$

$$\eta = d_1 / A^{1/2} = 2.54 \tag{43}$$

It is assumed that the metal deck offers lateral restraint;⁴⁷ consequently, buckling of the compressed members occurs only about their major axis. Using the interaction formula**

$$\frac{f_a}{F_a} + \frac{f_b}{F_b} \le 1.0 \tag{44}$$

with

$$F_b = 0.6F_y \tag{45}$$

and with

$$F_a = \left[1 - \left(\frac{KL_c}{r_x}\right)^2 \frac{1}{4\pi^2} \frac{F_y}{E}\right] \frac{F_y}{\text{F.S.}}$$
(46)

since $(KL_c/r) < C_c$, where K is equal to 0.7.† The Factor of Safety (F.S.) varies‡ and has a maximum value of 1.92 when $KL_c/r = C_c$. With $L_c = \sqrt{2}\lambda$:

$$A_{T,C} = \lambda^{2} \frac{q}{F_{y}} \left\{ \frac{K}{6\gamma^{2}} \frac{q_{f}}{q} \frac{L_{c}}{d_{1}} + \frac{C(256/\sqrt{2}\pi^{2})(\lambda/h) \left\{ (a^{2}/\lambda^{2})/[1 + (a^{2}/b^{2})] \right\}}{[1 - (K^{2}/\gamma^{2})(L_{c}/d_{1})(1/4\pi^{2})(F_{y}/E)]} \right\}$$
(47)

where

$$A_{T,C} = A_{T,C/\max}$$
 with $C = 2/\pi$

 $A_{T,C} = A_{T,C/avg}$ with C = 1

Equation (47) is solved by iteration (with the initial value of the length to depth ratio, L_c/d_1 , taken as 20), since d_1 is a function of A.

One notes that the average forces derived above are the average of forces that vary either as the function

$$f(x,y) = \sin \frac{\pi x}{a} \sin \frac{\pi y}{b}$$

- ** The interaction formula from the 1947 AISC Specification was used to simplify the program for preliminary design.
- † Naslund (Ref. 46) states: "It is the author's opinion that the joint rigidity created by the welded intersection of web and chord members is sufficient to permit the use of 70 percent of the distance between panel points for the chord member length."
- [‡] The actual factor of safety given in the 1963 AISC Specification varies: F.S. = $\frac{5}{3} + \frac{3}{8} \frac{(Kl/r)}{8C_c} - \frac{(Kl/r)^3}{8C_c^3}$

^{*} For a uniform load it was assumed that because of partial restraint $M = wl^2/10$. With the top chord longer than 16 ft an intermediate purlin was used and the moment increase was accounted for.

or as the first derivative of f(x,y) [Equations (2), (3), (4) and (7)]. When the cross-sectional area is derived from the force, the area varies in the same manner as the force for the tension members, and in approximately this manner for the other members. This, however, is not the actual variation at all points x and y, since the crosssectional area is also limited by the following code requirements, some of which reflect buckling:

For the compression diagonals (K = 0.7):

$$K \frac{L}{r_y} \le 200 \tag{48}$$

For tension members (bottom chord and tension diagonals, K = 1.0)*:

$$K \frac{L}{r_y} \le 240 \tag{49}$$

For the top chord, since it supports a flat roof:

$$\frac{d_1}{L_c} \ge \frac{f_b}{600,000 \text{ psi}}$$
(50)

From these restrictions the following *minimum* areas are derived:

Compression diagonal:

$$A_{X,C/\min} = A_{Y,C/\min} = \frac{\alpha\beta K^2 \lambda^2 [1 + (h^2/\lambda^2)]}{4 \times 10^4}$$
(51)

Tension diagonal:

$$A_{X,T/\min} = A_{Y,T/\min} = \frac{\alpha\beta K^2 \lambda^2 [1 + (h^2/\lambda^2)]}{5.76 \times 10^4}$$
(52)

Bottom chord:

$$A_{B,T/\min} = \frac{\alpha\beta K^2 4\lambda^2}{5.76 \times 10^4}$$
(53)

Top chord:

$$A_{T,C/\min} = \frac{2\alpha\lambda^2}{\gamma\eta} \sqrt{\frac{q_f(K)}{12,000 \text{ psi}}}$$
(54)

where the values of β , from Table 2, are:

Compression diagonal: $\beta = 3.90$ (55)

Tension diagonal: $\beta = 6.0$ (56)

Bottom chord: $\beta = 3.34$ (57)

The minimum area is further limited by the minimum area of rolled shapes available; these are shown boldface in Table 2.

The resulting variation in the cross-sectional area over the plan area of the truss is shown in Fig. 2. The minimum area occurs over region **P**. The point of maximum area occurs at the midpoint of the space truss for the top and bottom chords and at the center of opposite sides for the diagonals spanning in that direction. Let us now define an adjusted average area \bar{A} which is the average cross-section area of the members in region **Q** of Fig. 3. With either A_{\min} and the area of region **P** known, or \bar{A} and the area of region **Q** known, the weight of the structural member in the region under question can be determined from the equation:

Weight =
$$(A_{\min} \text{ or } \bar{A}) \frac{(\text{Area of region})}{(2\lambda)^2} L_m \times w_s$$
 (58)

where L_m is the total length of the member in a single pyramid element and w_s is the unit weight of steel. From these weights for the top chords, bottom chords and diagonals, the total weight carried by the truss may be calculated. If this weight is not within 5 percent of the assumed weight, the analysis and design is repeated. In addition, the weights of the members in these regions are used directly to determine the cost of the space truss.

Let us next determine the length to depth ratios for which the maximum allowable deflection is not exceeded. With the live load deflection limited by



Fig. 2. Variation of cross-sectional area plotted over ¼-span of the space truss



Fig. 3. Surface area per ton vs. cross-sectional area

^{*} The bottom chord is bolted; consequently, it is considered as pinconnected.

$$\Delta_{LL} \le \frac{b}{m}$$

m = 360 for floors
n = 240 for roofs

one obtains from Equations (8) and (38):

$$\frac{b}{h} \le \frac{\pi^2 q E \sqrt{2} (A_T / A_B)}{1.2 m q_{LL} F_y [1 + \sqrt{2} (A_T / A_B)]} \left(\frac{b}{a} + \frac{a}{b}\right)$$
(59)

For the problems considered in this work $A_T/A_B =$ 1.35, $F_y = 50$ ksi, $E = 30 \times 10^3$ ksi, m = 240, and

$$\frac{b}{h} \le 13.5 \; \frac{q}{q_{LL}} \left(\frac{b}{a} + \frac{a}{b}\right) \tag{60}$$

When this inequality is satisfied, stress and not deflection governs for design.

COST ANALYSIS

From the formulas of the previous section, the areas and quantities of material for each of the members of the truss may be determined. Since the ultimate goal of this study is the evaluation of minimum cost for various spans and live loads, it is advantageous to program these formulas for the digital computer. With this information the optimum span to depth ratio and span to grid spacing ratio can be determined by varying these parameters until the minimum cost is obtained.

The total cost of the truss was determined by considering the cost of the metal deck and the cost of the truss separately with the latter divided into cost of materials, fabrication, erection and painting.

By considering various open beam long span steel deck sections with and without intermediate purlins, the conditions for minimum weight of the metal deck was found. This resulted in the following equation for total unit weight of the steel deck and purlins (w_{rs}) in psf.:

$$w_{rs} = 2 + 0.0785L_c + 0.025(q_{LL} - 20)$$
(61)

where

 L_c = length of the top chord (ft)

 $q_{LL} = \text{live load (psf)}$

The cost of the roofing surface was taken as \$520.00 per ton. The weight of the built up roof and insulation was taken as 7 psf.

The material cost for members of the truss were obtained from a supplier for 36 ksi and 50 ksi steel. This data is summarized in Table 3 and in Fig. 4.

Fabrication costs were obtained from two fabricators, according to the details of Fig. 1 and to the following assumptions:



Fig. 4. Cost per ton vs. cross-sectional area

- 1. The pyramid elements would be either shop assembled or welded in a jig at the site.
- 2. The bottom elevation of the truss was assumed to be 60 ft above grade. Scaffolding would be built to this elevation or the elements would be assembled on the ground and lifted to the top of the columns.
- 3. The individual elements would be joined together by field bolting.

The resultant fabrication costs are summarized in Table 3 and Fig. 4.

The erection costs were taken as \$212.00 per ton for elements weighing less than 1.15 ton and \$162.00 per ton for elements weighing more than this.

Painting costs were figured at \$10.00 per ton for the shop coat, while the field coat was based on the surface area to be covered. The relationship between the surface area per ton and cross-sectional area for rolled shapes is given in Table 3 and Fig. 3. Field painting costs were then calculated on the assumption that a structural steel painter costing \$7.00 per hour could cover 1200 sq ft in an eight hour day, while one gallon of paint costing \$8.25 can cover 450 sq ft.⁴¹

		Member Designation	Area (in.²)	$Material Cost (\$/Ton)$ $F_y = 36 \text{ ksi} \qquad F_y = 50 \text{ ksi}$		Fabrication Cost (\$/ton)	Painting Field Coat Area(ft ²)/ton
	Com- pression	4WF13 6WF25 8WF48 8WF67	3.82 7.37 14.11 19.70	141 135 128 128		624 480 300 180	307 242 169 124
Diagonals	Tens. & Comp.	10WF 89 10WF 112 12WF 161 14WF 237	26.19 32.92 47.38 69.69	127 127 126 126	154 154 153 168	170 110 102 68	115 93 78 66
	Tension	4I7.7 6I12.5 8I23 12I50	2.21 3.61 6.71 14.57	143 139 137 133	150 146 154 154	624 624 480 182	404 337 232 149
Top Chord Bottom Chord		$\begin{array}{c} 2-5 \times 3 \times \frac{3}{8} \swarrow \\ 2-6 \times 3\frac{1}{2} \times \frac{3}{8} \swarrow \\ 2-7 \times 4 \times \frac{3}{8} \swarrow \\ 2-8 \times 4 \times \frac{1}{2} \swarrow \\ 2-9 \times 4 \times \frac{9}{16} \swarrow \\ 2-9 \times 4 \times \frac{3}{4} \swarrow \\ 2-9 \times 4 \times 1 \swarrow \\ 2-9 \times 4 \times 1 \swarrow \\ 2-15 \sqsubseteq 50 \\ 2-18 \varlimsup 58 \end{array}$	5.726.847.9611.5014.0018.3824.0029.2833.96	131 131 130 130 129 129 129 129 131 131	138 138 137 147 146 146 156 148 148	148 108 104 102 102 98 94 90 86	274 271 270 204 181 141 106 145 148
		ST 5-14.5 ST 8-32 ST 9-52.5 ST 10-71 ST 13-88.5 ST 16-110 ST 18-130 ST 18-150	$\begin{array}{r} 4 . 27 \\ 9 . 40 \\ 15 . 43 \\ 20 . 88 \\ 26 . 05 \\ 32 . 37 \\ 38 . 28 \\ 44 . 09 \end{array}$	148 138 138 142 140 141 141 141	165 155 155 159 157 158 168 168	$560 \\ 450 \\ 296 \\ 176 \\ 170 \\ 110 \\ 106 \\ 104$	250 172 133 112 104 98 89 78

Table 3. Cost vs. Cross-Sectional Area

a Available with a maximum yield stress of 46 ksi

With the above information included in the program, the computer results were checked against those from a simply supported space truss built in 1963. This comparison appears in Table 4. One should first note that, for the same live load, the dead load of the truss from the computer design is approximately two-thirds the weight of the 1963 space truss. This is a consequence of the different framing system which results in half as many diagonals as well as shorter compression elements. Since the Takenaka truss was unknown in 1963, such a saving in weight could not have been anticipated. As a consequence of the different truss weight, a comparison between total cost in dollars per sq ft would be meaningless. If, on the other hand, one compares cost in dollars per pound of steel, it is found that when these trusses carry the same live load the computer results give \$0.30 per lb, while the 1963 truss (allowing for a 10 per

cent increase for inflation) gives \$0.28 per lb. This difference can be explained by the fact that the members of the computer truss are lighter and, consequently, the unit cost in dollars per pound of steel is greater. If, on the other hand, one compares trusses that have approximately the same weight, one finds that the costs of the trusses are identical (figures in boldface).

It must be recognized that the costs in this study will vary with time and geographic location. In order to make adjustments possible, these costs are broken down into two parts: one tied to general costs, which will increase with time as general construction costs increase, and one tied to a specific base which is dependent upon both geographic location and time. This division is given in Table 5. It must be recognized that this division is in part arbitrary and any adjustments made can only be approximate.

T.	Space Truss	Compute	Computer Design		
Item	Built in 1963	Same Live Ld.	Diff. Live Ld.		
Dimensions: a b h λ	300 ft 400 ft 17 ft to 30 ft 16.67 ft	300 ft 400 ft 15 ft 16.67 ft	300 ft 400 ft 25 ft 16.67 ft		
Weight of Roofing: Roofing Material Insulation Acoustical Deck Lighting Joists	2.5 psf $\frac{1}{4}$ $\frac{1}{2}$				
	10.5 psf	9.8 pst	10.3 pst		
Weight of Truss	15.0 psf	9.3 psf	13.5 psf		
Total Dead Load	25.5 psf	19.1 psf	23.9 psf		
Live Load	20 psf	20 psf	40 psf		
Live Load Reduction Factor	0.6	0.6	1.0		
Cost of Truss: \$/sq ft (1963) \$/sq ft (1967) \$/lb (1967)	\$3.75/sq ft \$4.13/sq ft \$ 0.28/lb	\$2.85/sq ft \$0.30/lb	\$3.82 sq/ft \$ 0.28/lb		
Framing Systems: Top Chord Bottom Chord Diagonal	b 1963 Space Trus		b Ater Design		

Table 4. Comparison between the Computer Design and a Space Truss Built in 1963.



Fig. 5. Weight of truss vs. span

Table 5					
Item	Tied to General	Tied to Specific Costs			
nem	Costs (%)	%	Rates Used in This Study		
Metal deck		100	Price @ \$520.00/ton		
Material		100 Basic price @ \$140.00/ton structural shapes			
Fabrication	40	60	Welder @ \$6.50/hr		
Erection	40	60 Steel erector @ \$6.50/hr			

	(Max. value of 2A minited by $2A = 0.20$ and $\sqrt{2} A \leq 50.0$ ft)						
	b	2λ max (ft)	$(b/2\lambda)$ min	Other values of $b/2\lambda$ used			
	100	20	5	10			
	200	40	5	10, 15			
	300	30	10	15			
l	400	40	10	15, 20			
	500	33.3	15	20			
	600	40	15	20			

Table 6. 2λ and $b/2\lambda$ Used in Study (Max, value of 2λ limited by $2\lambda = 0.2b$ and $\sqrt{2}\lambda \le 30.0$ ft)

RANGE OF GEOMETRIC PARAMETERS $a/b, b, b/2\lambda$ and b/b

In the cost analysis, aspect ratios (a/b) of 1.0, 1.2 and 1.4 were considered with the span *b* varying from 100 to 600 ft in increments of 100 ft. With an aspect ratio of 1.2, the largest possible grid spacing (2λ) is 0.2*b*. The grid spacing is further limited by the maximum span of the metal deck on the top chord $(\sqrt{2\lambda})$ taken as 30 ft. These restrictions taken together result in the maximum values of 2λ indicated in Table 6. Any smaller grid spacing must be a subdivision of this maximum value. Much greater flexibility exists in the choice of $b/2\lambda$ for aspect ratios of 1.0 and 1.4; however, for purposes of comparison the same values were used for each of the aspect ratios.

Alternatively, the span to depth ratio b/h may be varied at will. Naslund in dealing with a different type of space truss has suggested a value of 20,46 while Gensert in discussing one way trusses has suggested a value of 10.14 For the purposes of this study values of b/h from 10 to 30 were considered in increments of 2.5.

DISCUSSION AND CONCLUSIONS

The computer program was used to solve space trusses defined by the aforementioned parameters and loaded with live loads of 20 and 40 psf. With these loads and the range of parameters noted above, a total of 756 different trusses were solved.

An attempt was made to abstract from the resulting data general information independent of live load and aspect ratio. It was possible, for example, to relate the minimum total cost in dollars per sq ft to the weight of the truss (Fig. 6). The weight of the truss in turn is related to the span, loads and aspect ratios through Fig. 5. From Figs. 7 and 8* the cost of the metal deck can be determined. Subtracting this from the total cost one finds the cost of the truss. With this cost and the



Fig. 6. Minimum total cost per sq ft (including cost of metal deck and purlins) vs. truss weight per sq ft



Fig. 7. Length of top chord member per minimum cost



Fig. 8. Cost of metal deck (@ \$520/ton)



Fig. 9. Costs as a percent of truss cost

weight of the truss known, the fabrication, erection, material and painting costs can be found from Fig. 9. By the method previously discussed, which uses the information from Table 5, these costs can be adjusted to account for different labor costs and inflation.

^{*} The dip in the curve at b = 500 ft is due to the fact that the maximum value of 2λ considered for the 500-ft span is less than that considered for either the 400 or 600-ft span. This is a consequence of the restriction placed on 2λ as noted in the previous section and in Table 6.



Fig. 10. Percent increase in total cost as a function of b/2 and b/h. (w = 40 psf, a/b = 1.0)

It was observed that for trusses weighing from 4 to 6 psf, the minimum area of rolled shapes instead of stress governed the size of members. As a result the weight of the truss and, consequently, the cost were less dependent on the live load and aspect ratio (Fig. 5). In this range of weight per sq ft the relationship between fabrication, erection and material costs remained constant (Fig. 9). since generally the members of the different trusses were identical. In order to maximize the use of the crosssectional area available, the most economical design resulted in fewer elements and larger slenderness ratios $(b/2\lambda = 5 \text{ and } b/h \text{ from 10 to 15, Fig. 10})$. It would seem that for spans up to 100 ft one should not consider this type of truss except for much heavier loads, and that for the lighter loads considered in this work a truss built up from tubular sections or specially rolled shapes should be considered instead.

One notes that with the weight of the truss varying from 6 to 20 psf, the fabrication cost decreases rapidly in comparison with the erection and material costs (Fig. 9). This corresponds to the relationship shown in Fig. 4 for cross-sectional areas less than 30 sq in. This fact, plus the actual observation of the computer output for these problems, confirmed that the members required fall within the range of those available for rolled shapes and reflect design requirements based on stress. For the span and live load for which these truss weights are applicable, i.e., for spans of 200 to 400 ft and for a live load of 40 psf (Fig. 5) for the square truss,* it is seen that a variation of the span to depth ratio from 12 to 22 results in less than a 4 per cent increase in the minimum cost (Fig. 10).



Fig. 11. Minimum total cost per sq ft vs. span

This may be explained by the interplay between the weight of the chords and the weight of the compression diagonals, where the chord forces and, hence, the chord area decrease linearly with increase in depth, while the area of the compression diagonals increase as the square of the depth because of elastic buckling. The increase in length of the diagonals also adds to their increase in weight.

For trusses weighing more than 20 psf one notes that material costs exceed erection costs while for lighter trusses they did not (Fig. 9). This is a reflection of the fact that for heavier trusses many members must be built up from plates and an increment increase in cost (15 percent) was added to the usual material, i.e., member cost. Otherwise the material and erection costs in dollars per ton are approximately constant.

^{*} The curves and conclusions for other aspect ratios and loads are similar. The number of curves in each family (i.e., the variation of $b/2\lambda$) is limited by the restrictions noted previously.

The above comments based on the weight of the truss may be related to span, live load and aspect ratio through the use of Fig. 5. The actual relationship between these parameters and minimum cost is summarized in Fig. 11.

The above relationships might possibly be used to estimate the cost of trusses involving other framing systems once the weight of the truss is known. For example, using the 1963 truss of Table IV as a case in point, one obtains a total cost of \$5.12 per sq ft (Fig. 6). From Figs. 7 and 8 the cost of the metal deck and purlin is found to be \$0.96 per sq ft. This leaves a cost of \$4.17 per sq ft for the truss which compares with an adjusted 1967 price of \$4.13 per sq ft.

For all the problems considered in this work, steel with a yield strength of 36 ksi was used for the compression diagonals which are designed for elastic buckling. On the other hand, high strength steel (50 ksi) generally proved most economical for the top chords, bottom chords and tension diagonals. An exception occurred for trusses weighing less than 6 psf, for then minimum area instead of stress governed the size of members.

The grid spacing and span to depth ratios considered have been noted. Altogether a total of 756 different trusses were solved. In general the optimum grid spacing proved to be the largest permitted (see Table VI). The cost was much more sensitive to grid spacing than to the span to depth ratio. The optimum span to depth ratio varied from 10 to 20 with less than a 10 percent variation in cost in this range.

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